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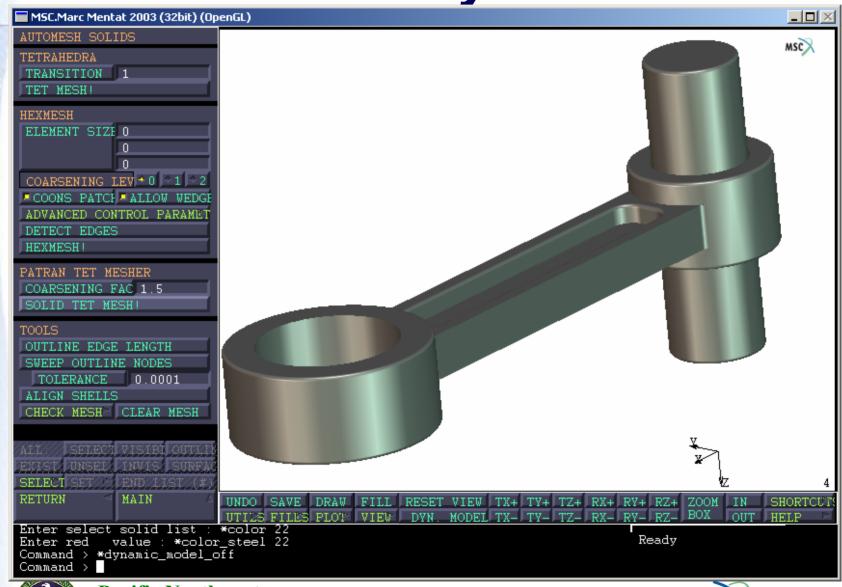
- Meshing/Remeshing
- Analysis of Composite Materials (shell/solid)
- Fracture Mechanics Capabilities
- Heat Transfer and Coupled Thermal Stresses
- Running Jobs in Parallel
- Fluids
- User Subroutines
- Future Work
- PEN Fuel Cell Modeling







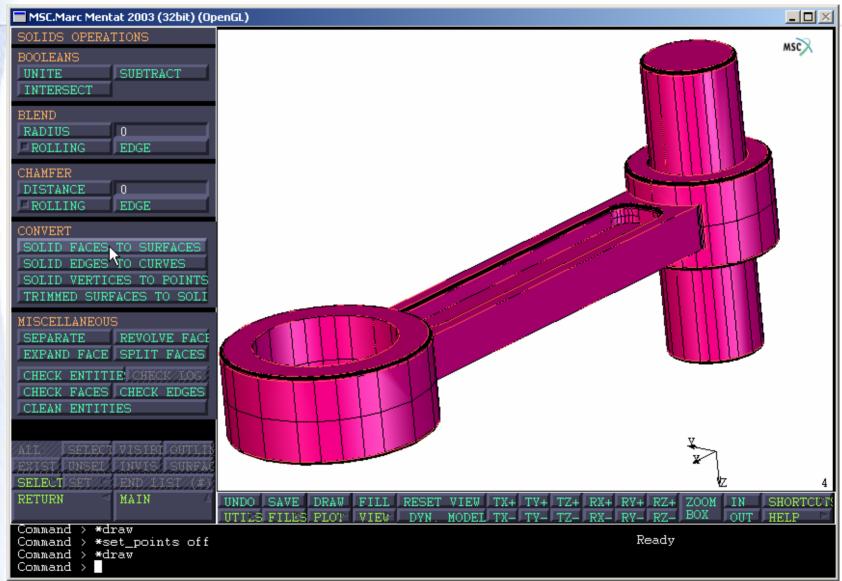
CAD Connectivity: ACIS Solid







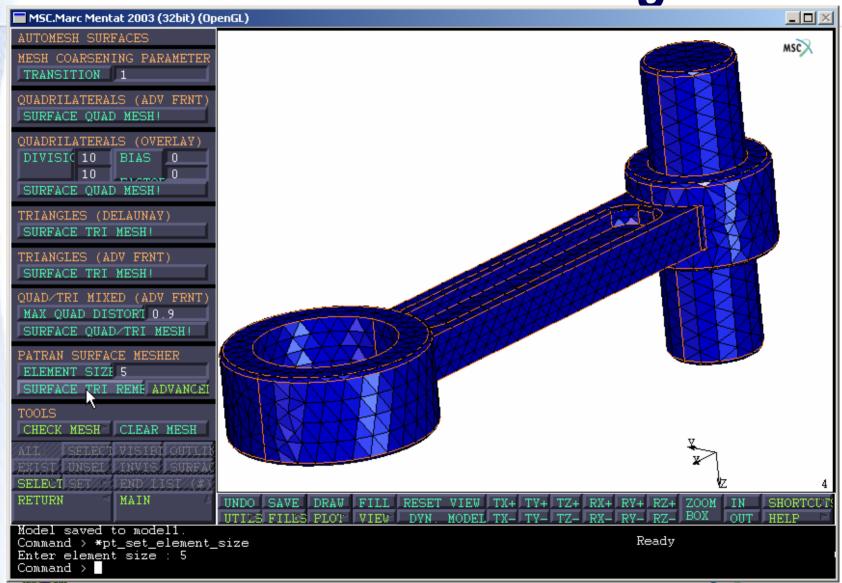
Convert to Surfaces







Automatic Meshing

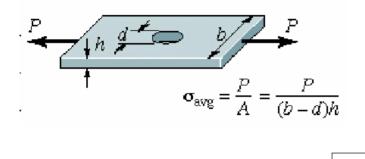






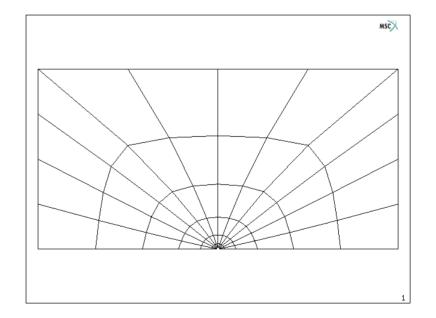


Uniform Axial Load



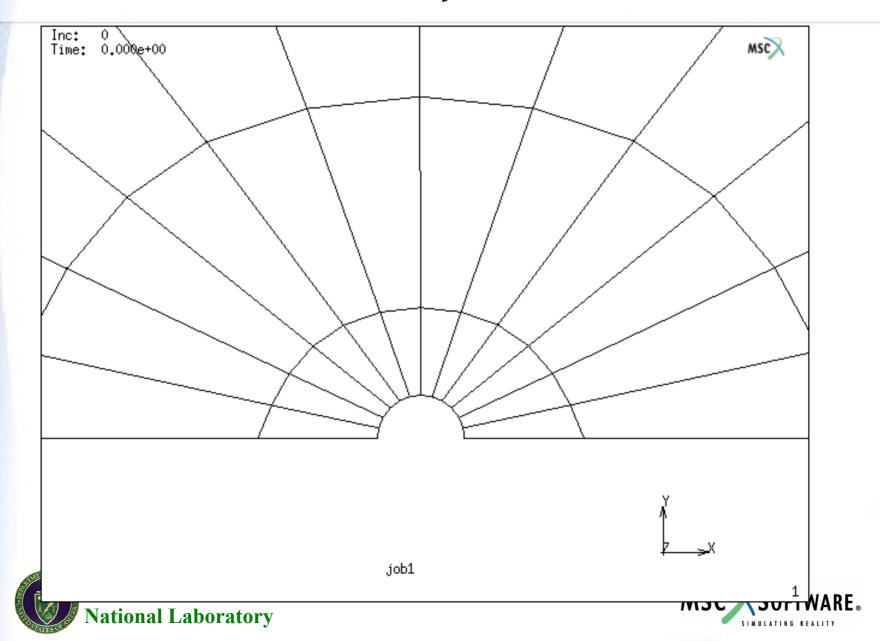
$$\sigma_{\text{max}} = K_c \sigma_{avg}$$

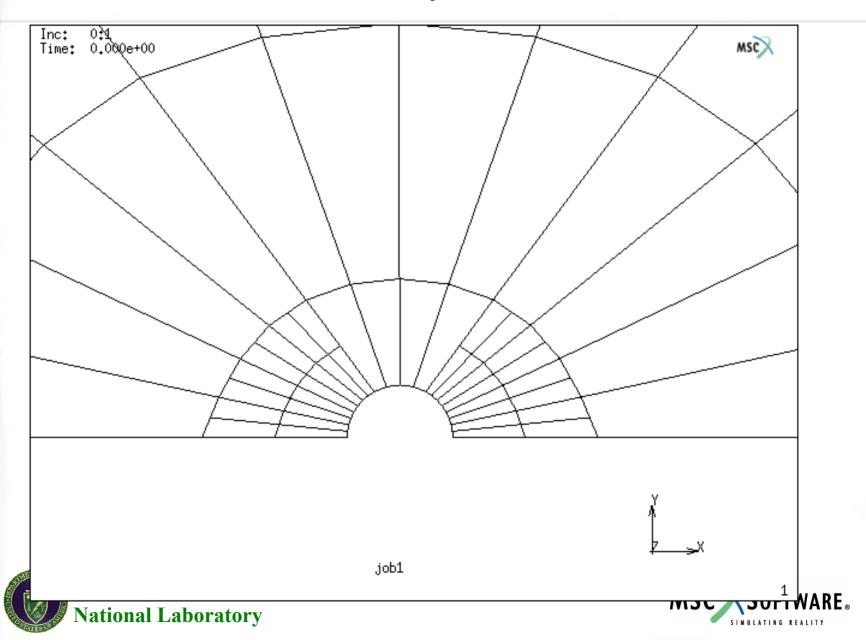
$$h = 1$$
$$b = 10$$
$$d = .1$$

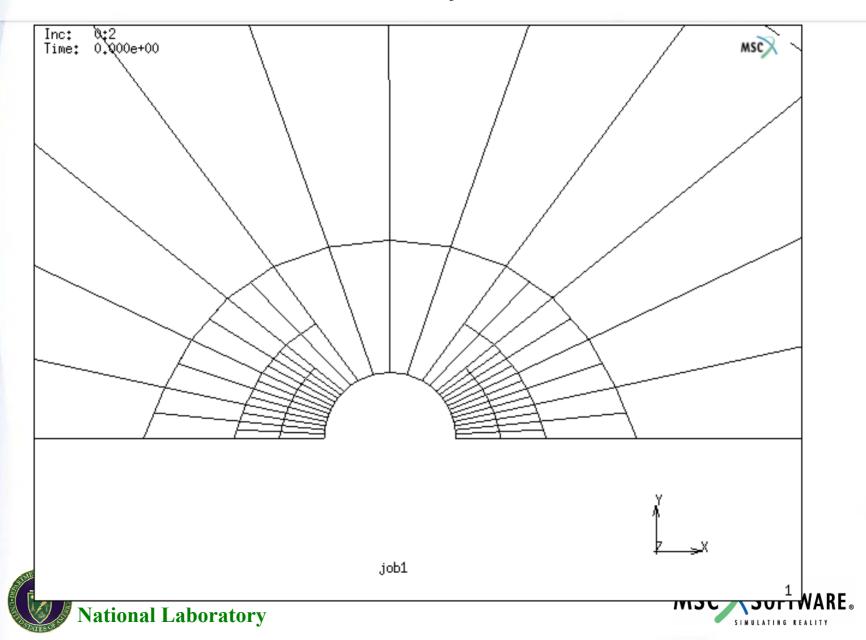


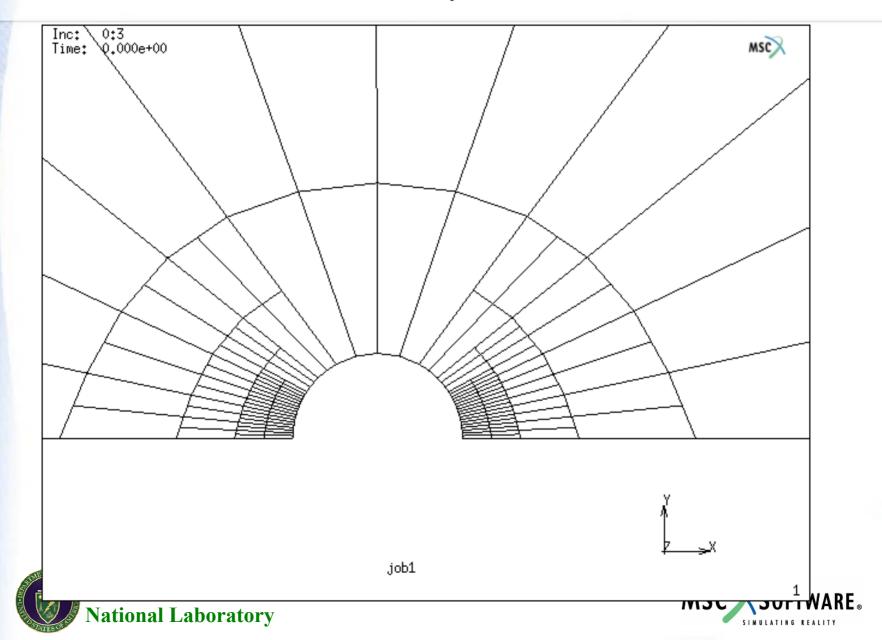


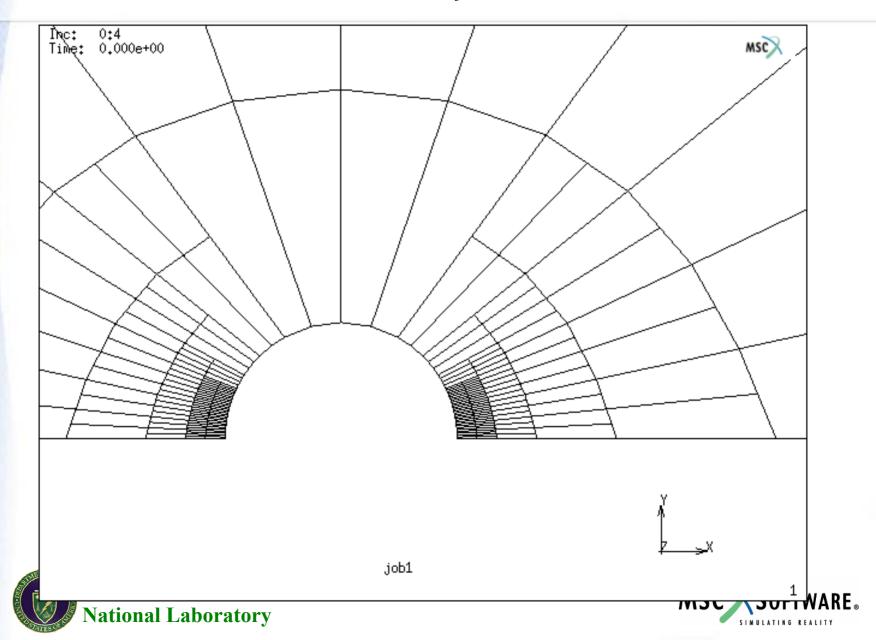


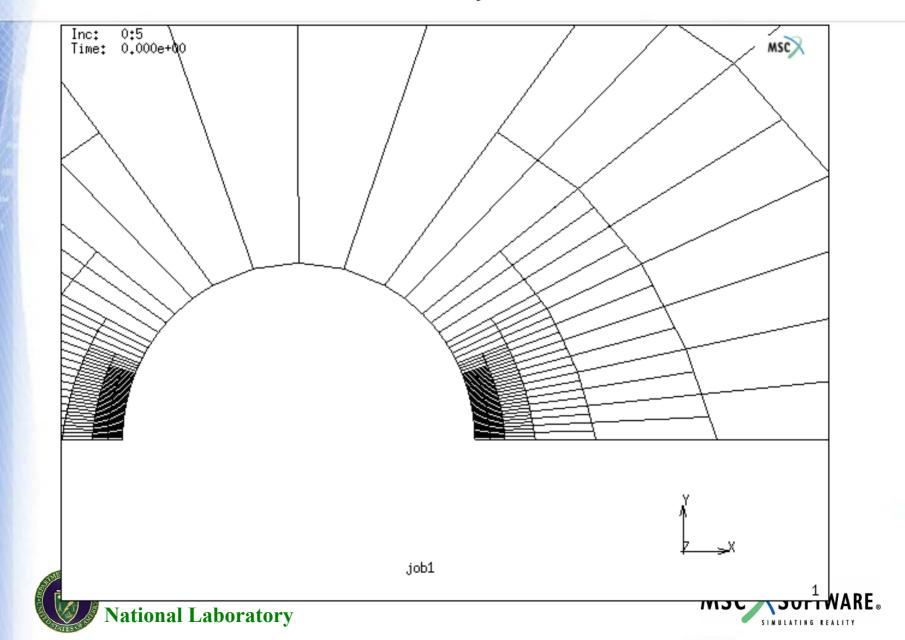


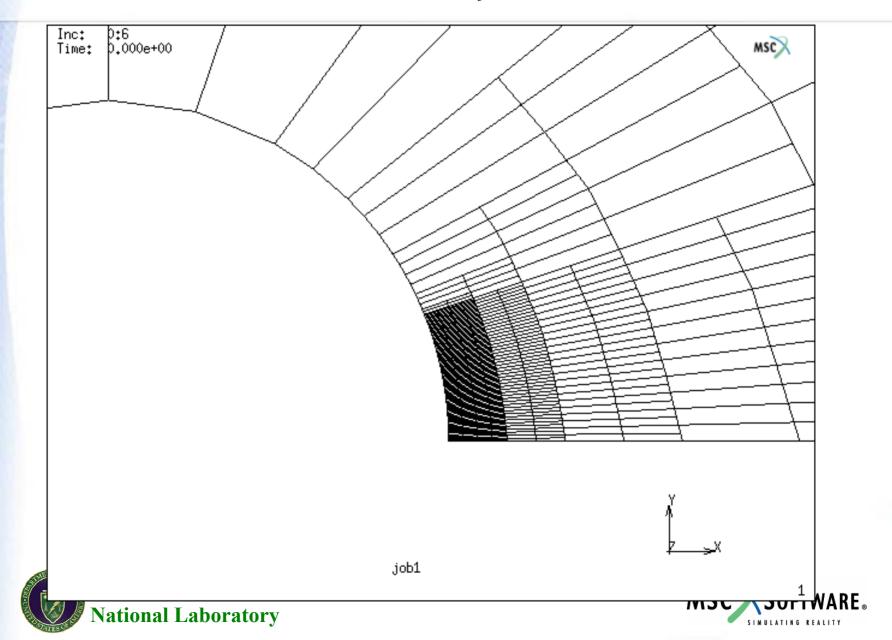


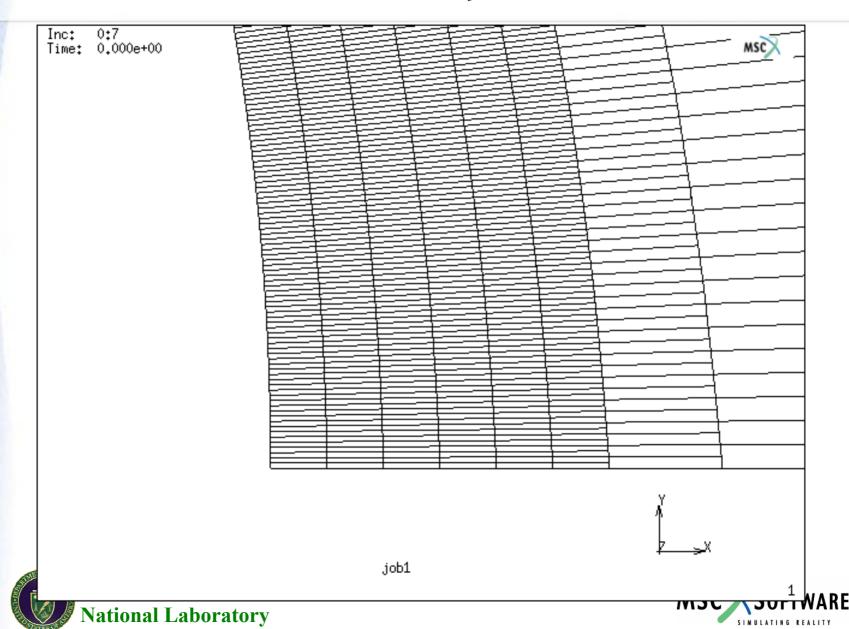






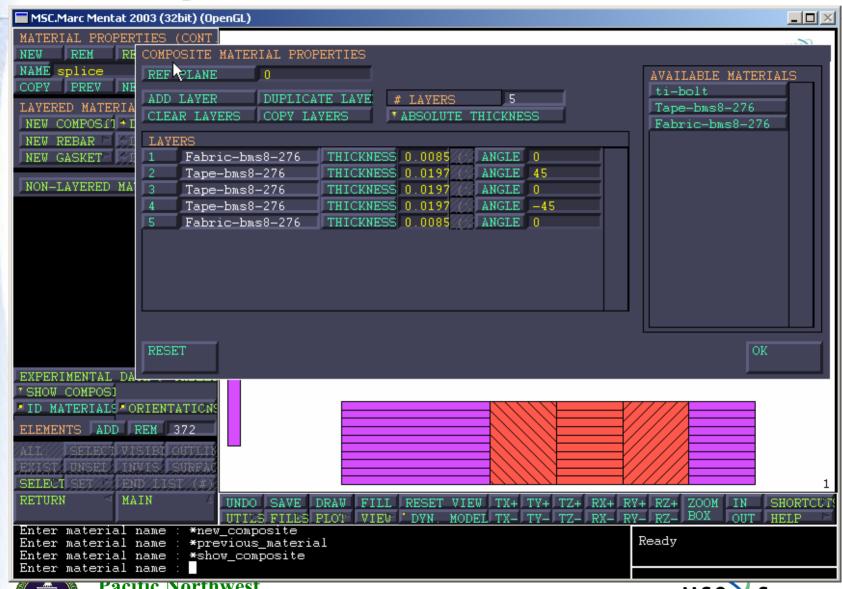








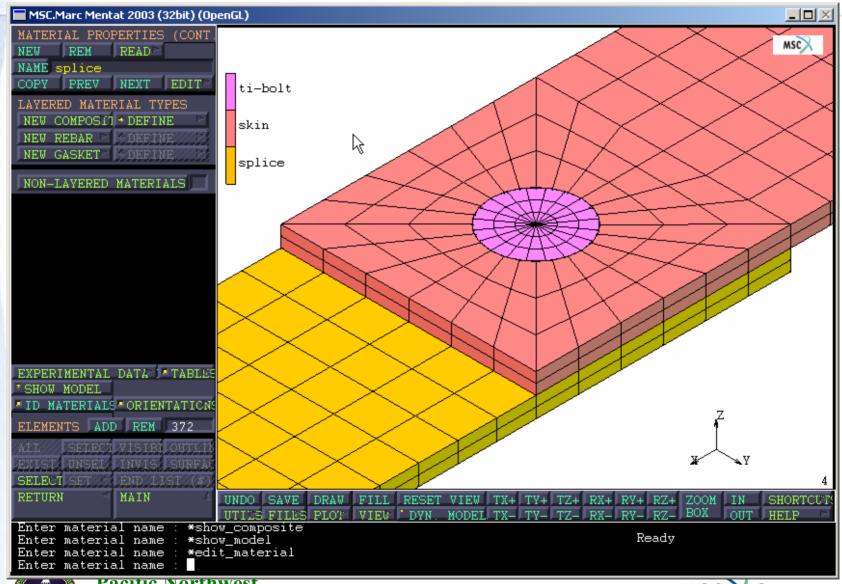
Composite Shell and Bricks







Composite Bricks

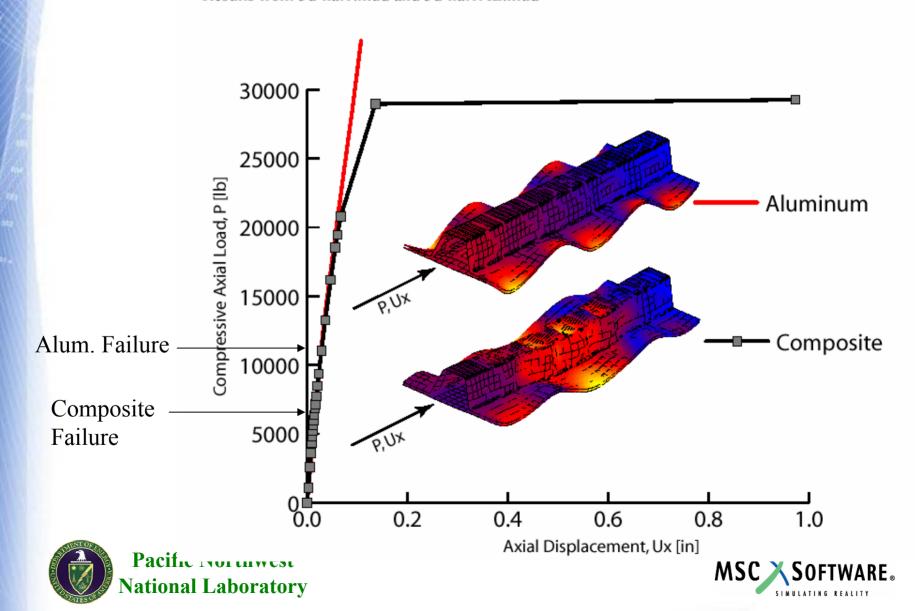


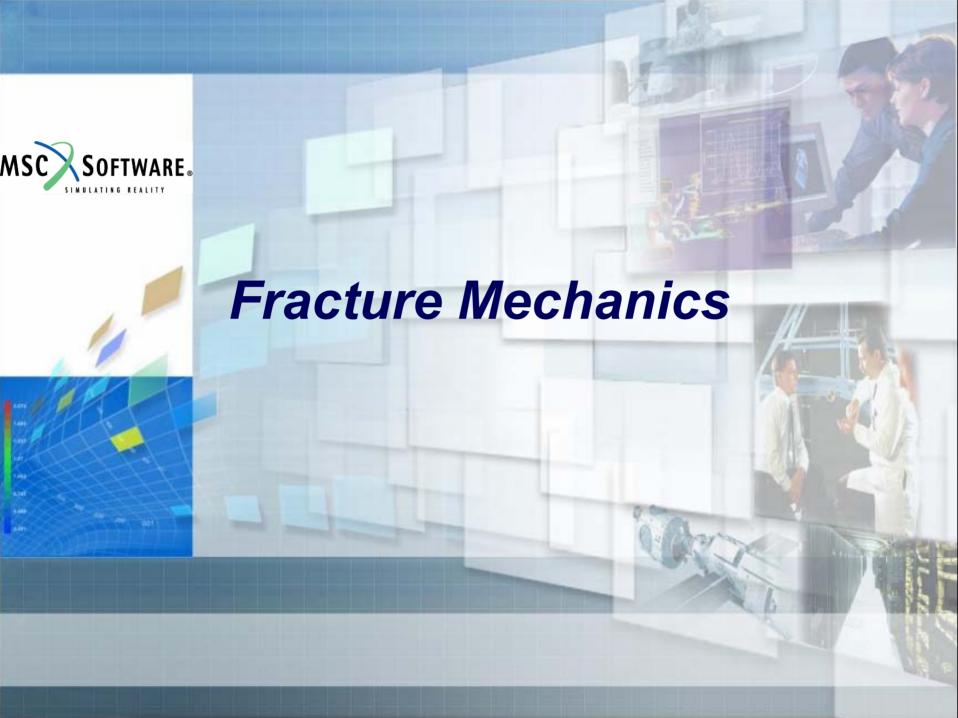




7E7 Mid Skin Buckling

Results from 3d-hat1.mud and 3d-hat1AL.mud





J-Integral Estimations

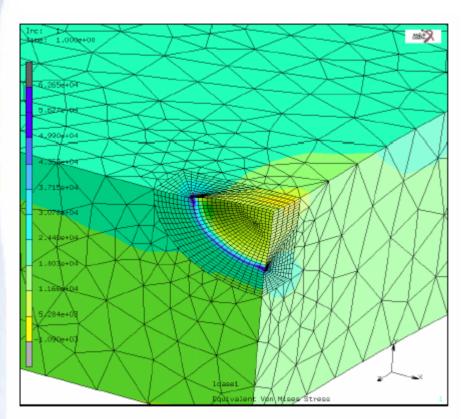


Figure 19-6 Equivalent von Mises Stress Contours

Table 19-1 J-integral Estimations

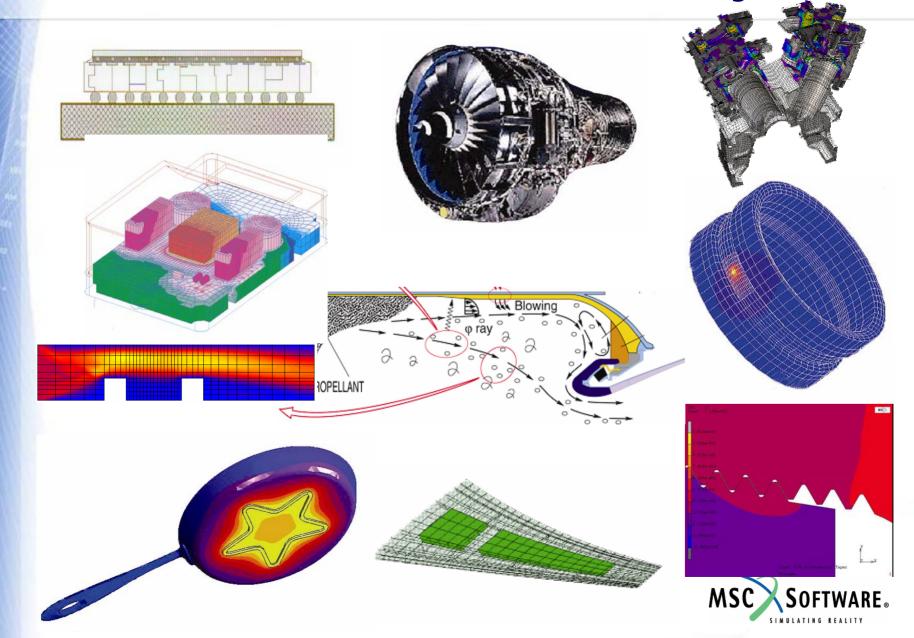
| Crack Tip Node | Path Radius | J-integral Value |
|-------------------|-------------|------------------|
| 1235 | 1.1180E-01 | 3.1508E+03 |
| 1235 | 2.2361E-01 | 3.2027E+03 |
| 1235 | 3.3541E-01 | 3.1918E+03 |
| 1235 | 4.4721E-01 | 3.1984E+03 |
| 1235 | 5.5902E-01 | 3.1961E+03 |
| 1235 | 6.7082E-01 | 3.1947E+03 |
| 1235 | 7.8262E-01 | 3.1934E+03 |
| | | |
| 3656 | 1.1187E-01 | 3.1553E+03 |
| 3656 | 2.2375E-01 | 3.2247E+03 |
| 3656 | 3.3562E-01 | 3.2335E+03 |
| 3656 | 4.4750E-01 | 3.2378E+03 |
| 3656 | 5.5937E-01 | 3.2338E+03 |
| 3656 | 6.7125E-01 | 3.2276E+03 |
| 3656 | 7.8312E-01 | 3.2216E+03 |







The World of Thermal Analysis



- Steady State Simulation
- Transient Simulation
- Temperatures can be easily used in uncoupled thermal stress analysis
- Coupled Thermal-Structural Analysis
- Coupled Thermal-Electric (Joule Heating)
- Coupled Thermal-Electric-Structural
- Choice of Time Step Procedures
 - Fixed Time Steps
 - Adaptive Time Steps





- 1-d
- 2-d (planar and axisymmetric solid and axisymmetric shells)
- 3-d (solid, and shells)
- Heat transfer shells may have any number of layers, temperature varies either linearly or quadratically through layer
- Nonlinear Transient Cyclic Symmetry
- All heat transfer elements have comparable structural elements for thermal stress analysis or coupled analysis





 Isotropic , Orthotropic or Anisotropic thermal properties.

 All properties may be function of temperature.

 Latent heat effects included to model phase changes





- Point or Distributed Fluxes
- Convective Boundary Conditions
- Radiative Heating
- View Factor Calculations efficiently done using Monte Carlo method
- Internal Heating due to plasticity or friction in coupled structural analysis.
- Thermal Contact in coupled structural analysis.





Applications

- Energy Industry
- Engines (Gas Turbine, Diesel, Automotive)
- Rockets
- Electronics
- Fire Safety
- Manufacturing
- Welding









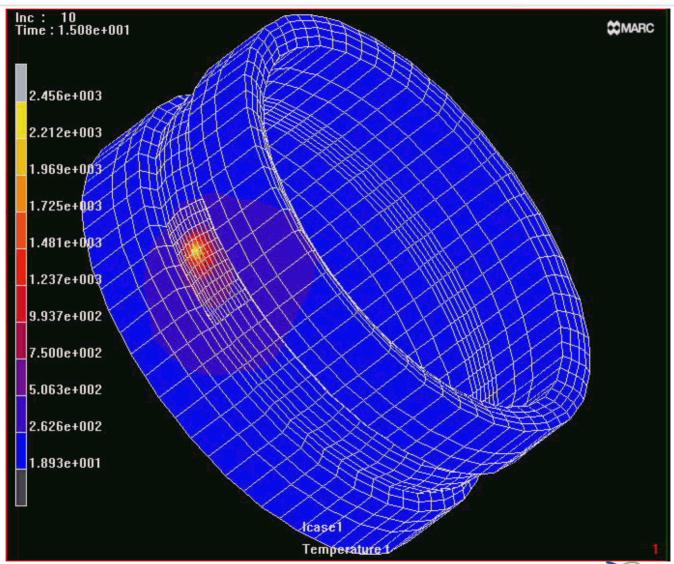








Welding Simulation







Channel

 Special Modeling techniques to represent fluid flow in channels

 Special modeling techniques to model convection and radiation across small gaps.





Computationally Efficient Solution Methods

Direct Sparse Solvers

Iterative Solvers

Parallel Processing using Domain Decomposition





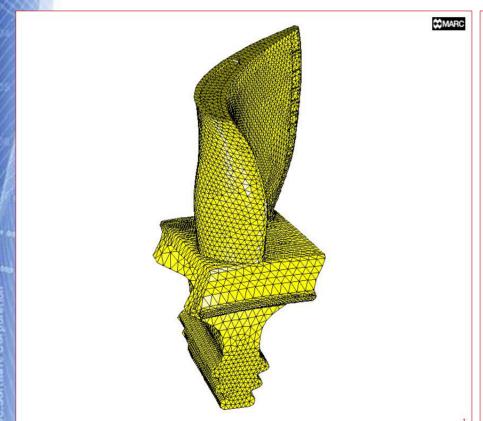
Turbine Blade Structural Analysis

- Number of Domains: 4
- Number of Elements: 72K
- Degrees of Freedom: 55K
- Scaling: 4.2X





Turbine Blade Structural Analysis









Thermal Contact

If dist < d1 then thermal conduction</p>

If d1< dist < d2 then simplified thermal radiation

If d2 < dist then no contact</p>





Near Contact

- Convection
- Natural convection
- Radiation
- Distance dependent convection
- Q = hcv*(T2-T1)+hnt*(T2-T1)ent +
- sigma*eps*(T24-T14) +
- (hct (hct-hbl)*gap/dqnear)*(T2-T1)





Thermal Contact Input

- Hct contact thermal coefficient
- Hcve environment thermal coefficient
- Tsink sink temperature
- Hcv near convective coefficient
- Hnc near natural convection coefficient
- Bnc exponent for natural convection
- Em emissivity
- Hbl lower limit of distance dependent convection coeff
- Dqnear distance below which near thermal behavior is applied





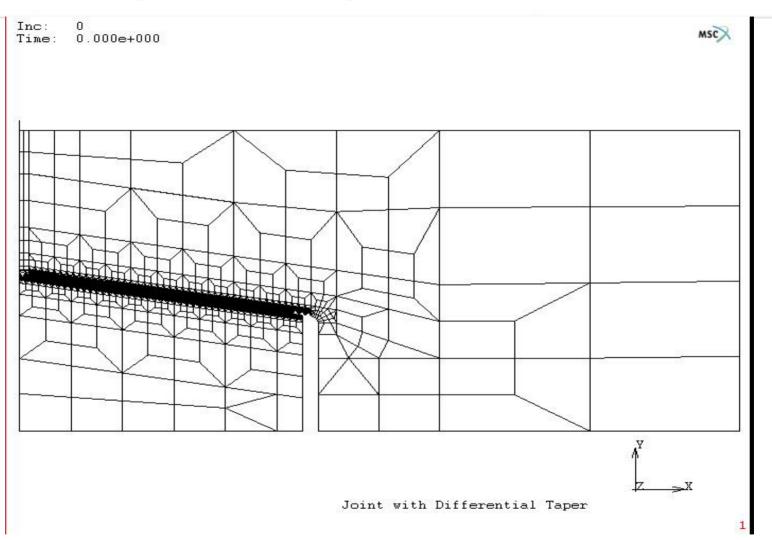
Coupled Joule Heating

- Weak coupling between mechanical, thermal, and electrical fields
- Typical application: high voltage electrical switch
- Electronic circuits





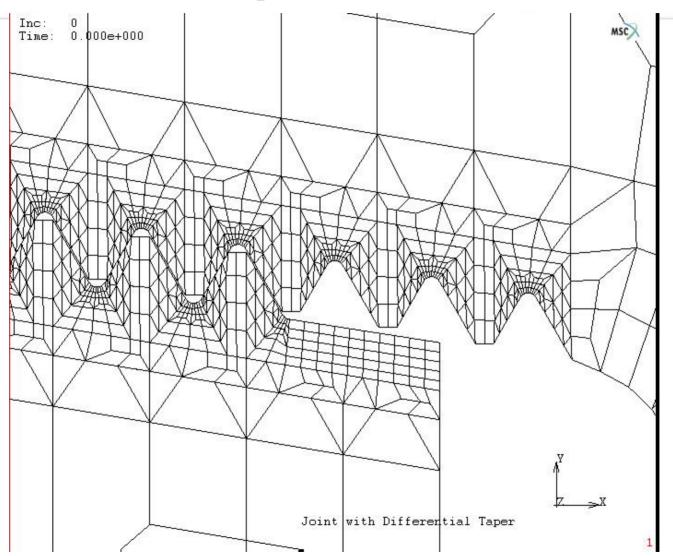
High Voltage Device







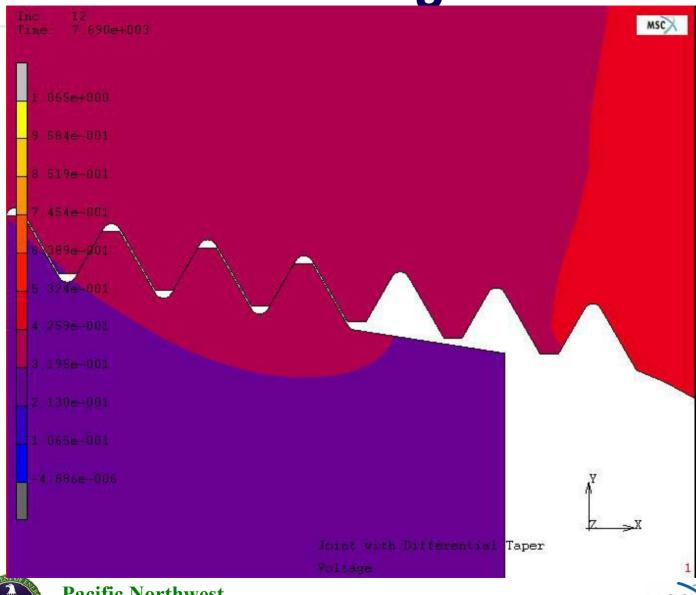
Close up of Threads







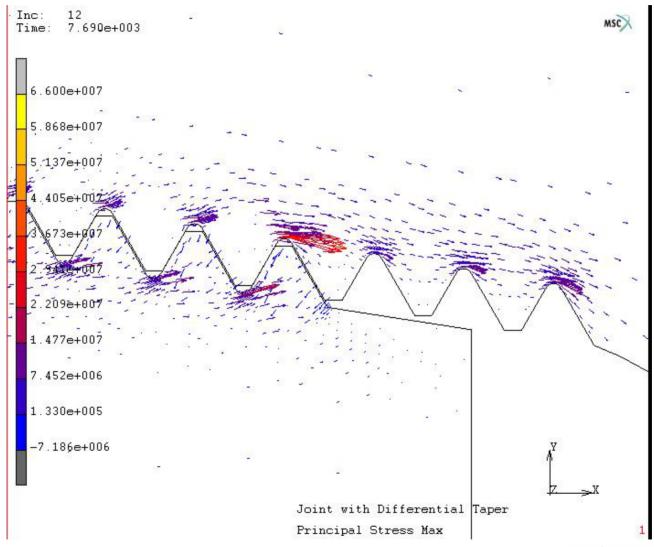
Voltage







Induced Stress Due to Contact and Thermal Strains







Fluid -Thermal Analysis

- Navier Stokes Finite Element Procedure Used
- Tightly Coupled Analysis
- Incompressible Fluid
- Laminar Flow
- Newtonian or Non-Newtonian Fluid
- 2D or 3D fluid flow with any type of element
- Forced or Free Convection





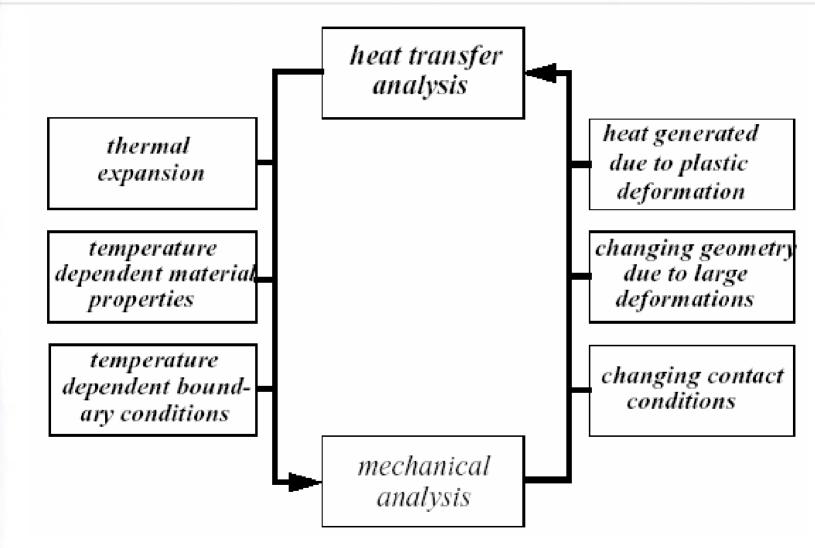
Fluid-Thermal Applications

- Electronic Packaging
- Quenching
- Fuel Cells





Coupled Thermal Stress





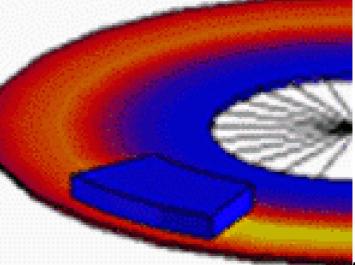


Disc Brake

Disc Brake
Dynamics
Coupled
Contact





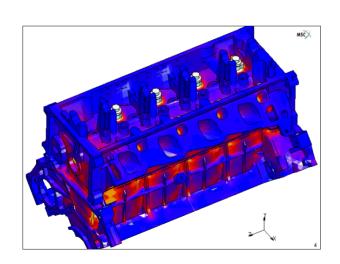


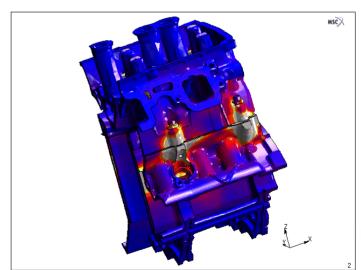


Super Scalable Parallel

| Number Of CPU's | 1 | 2 | 4 |
|-----------------|------|------|------|
| CPU Time (sec) | 6429 | 2491 | 1128 |
| Speedup | 1.0 | 2.6 | 5.7 |

Linux cluster of 4 HP workstation xL-class with two of Intel's® Pentium® III 1GHz processors and 4GB of RAM/CPU

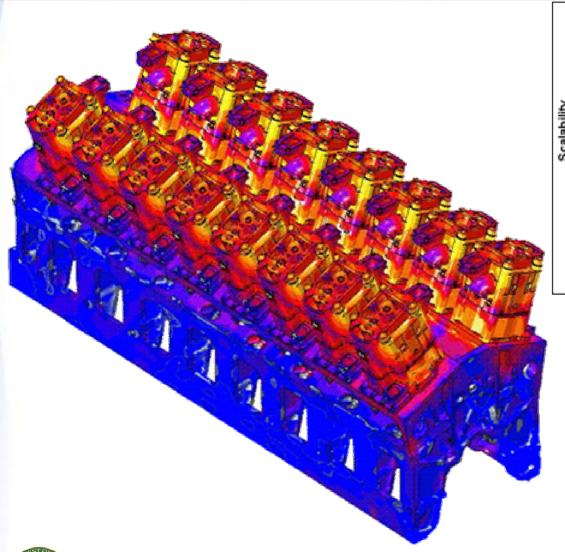


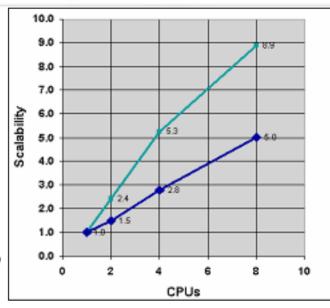






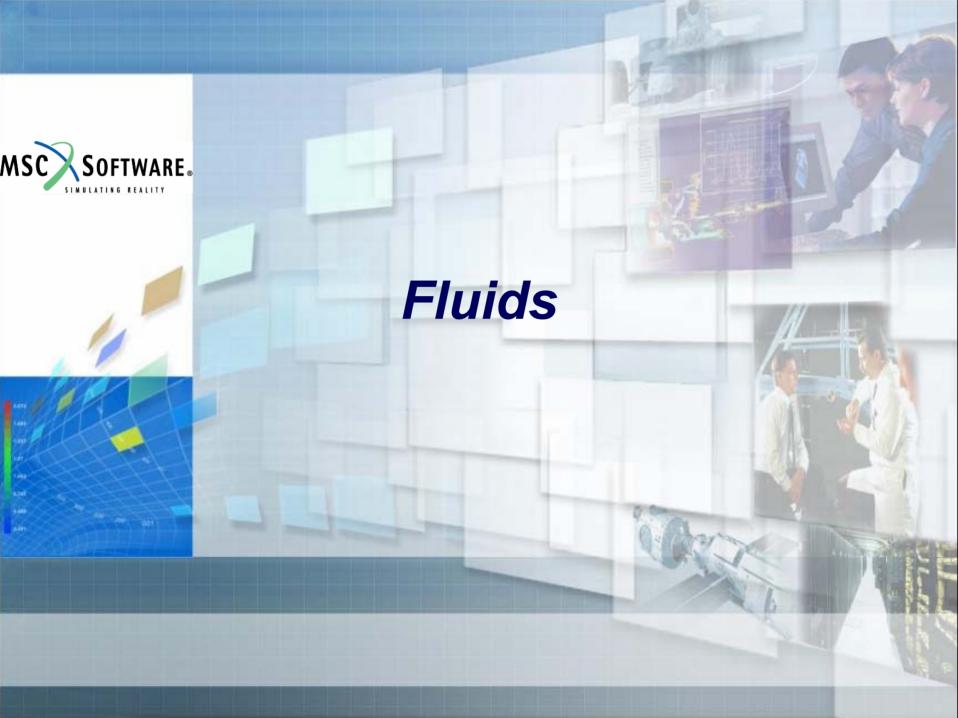
Parallel for Engine Assembles











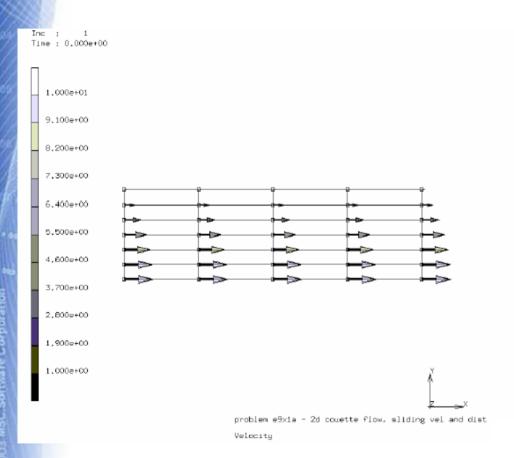
MSC.Marc Volume E: Fluid Demonstration Problems

| 9.1 | Planar Couette Flow, 9.1-1 |
|-----|---|
| 9.2 | Poiseuille Flow, 9.2-1 |
| 9.3 | Fluid Squeezed Between Two Long Plates, 9.3-1 |
| 9.4 | Driven Cavity Flow, 9.4-1 |
| 9.5 | Flow Past a Circular Cylinder, 9.5-1 |
| 9.6 | Flow Over Electronic Chip, 9.6-1 |
| 9.7 | Natural Convection, 9.7-1 |
| 9.8 | Flow Around Tubes, 9.8-1 |





Plot of the Couette Flow Velocity Field



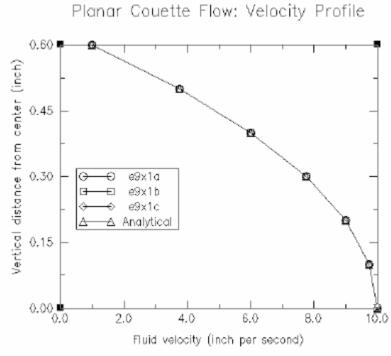
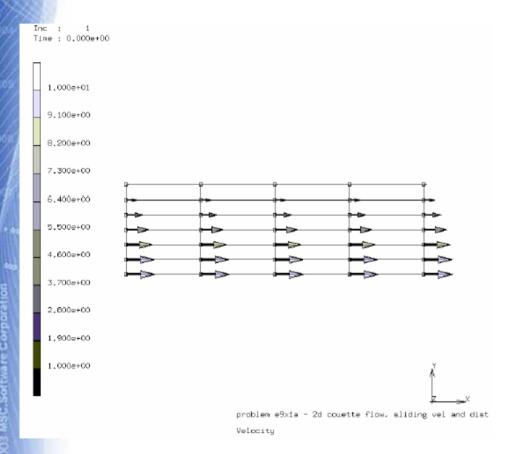


Figure 9.1-4 Comparison of Computation and Analytical Results





Plot of the Poiseuille Flow Velocity Field



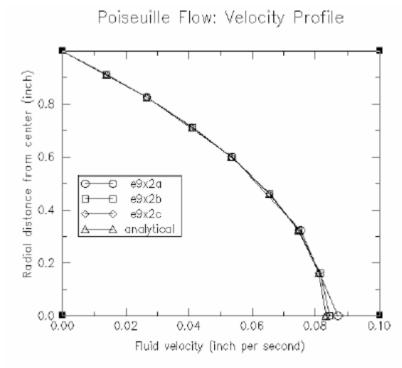


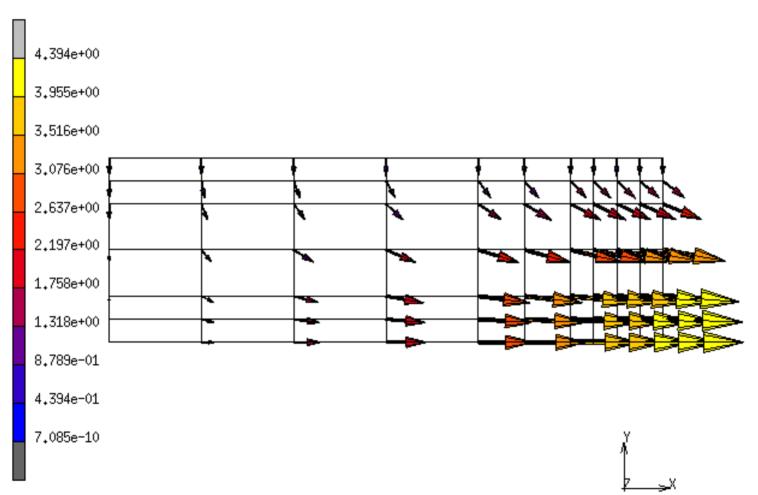
Figure 9.2-4 Comparison of Computation and Analytical Results





Fluid Squeezed Between Two Long Plates

Inc: 1 Time: 0.000e+00 MSC

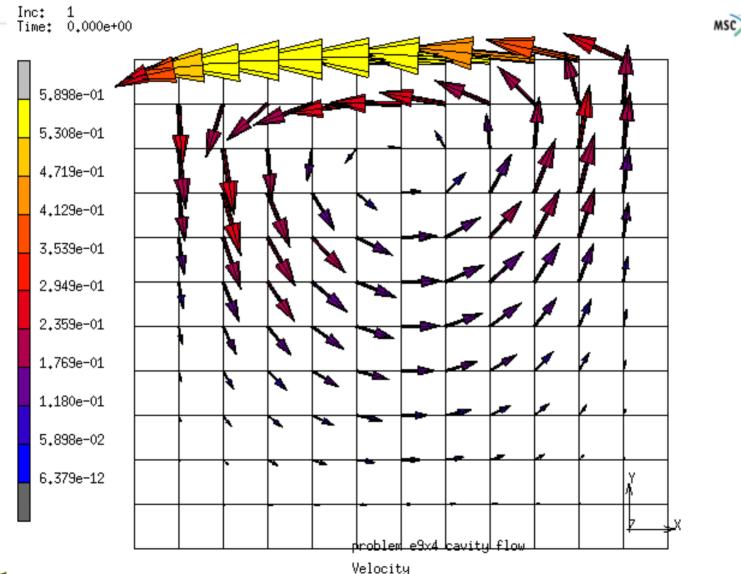


blem e9x3a: 2D fluid squeezed between plates, element type Velocity





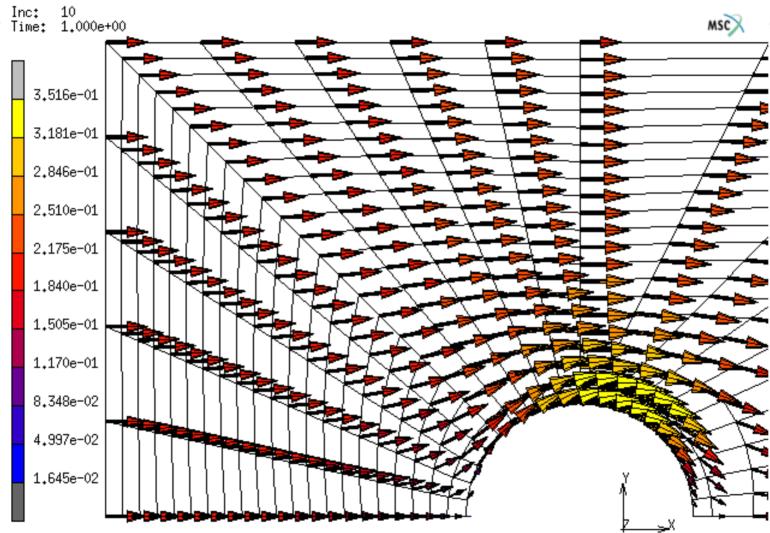
Vector Plot: Driven Cavity Flow Velocity Field







Flow Over Cylinder Transient Velocity Field



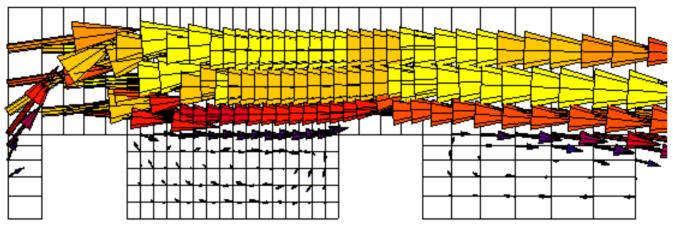
e9x5e - fluid around a cylinder, no 3rd (P) B_*C_* Velocity



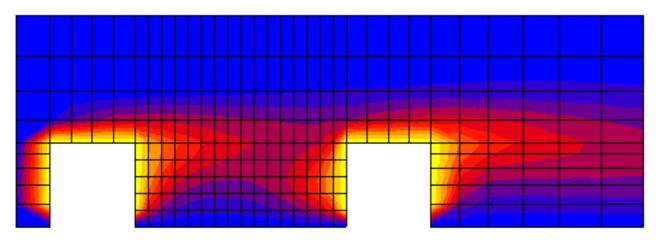


Flow Over Electronic Chip

Velocity Vector



Temperatue

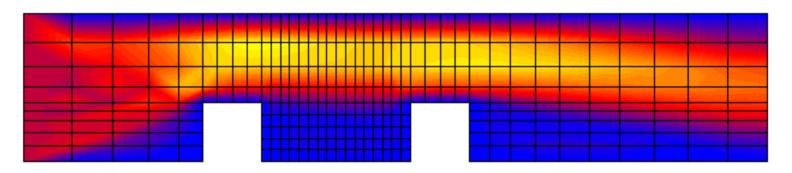




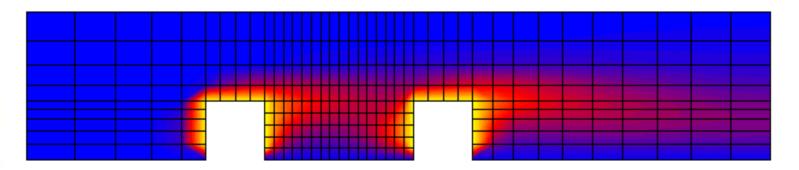


Flow Over Electronic Chip

Velocity Magnitude



Temperatue









Output Quantities Subroutines

| | User Subroutine | Required Parameters or Model Definition Options | | | Purpose | |
|---|--------------------|--|--|---|--|--|
| | ELEVAR | UDUMP | | Allows postprocessing of element results. | | |
| | ELEVEC | UDUMP | | Allows postprocessing of element results in harmonic analysis. | | |
| | INTCRD | | Mal | Makes available integration point coordinates. | | |
| | IMPD | UDUMP | Allows postprocessing of nodal vector results. | | ng of nodal vector results. | |
| | PLOTV | POST Defines element quantity to be written to | | antity to be written to post file. | | |
| | CBCINE | Dummy routine available at the begin each increment. | | lable at the beginning of | | |
| | UBGITR | _ | Dur | nmy routine avai | lable at the beginning of | |
| | UEDINC | | L | oads and | Boundary Con | |
| | UELOOP | | | User Subroutine | Required Parameters o Model Definition Option | |
| (| UPOSTV) | POST | | CREDE | THERMAL LOADS | |
| | | | | CUPLFX | COUPLE | |

Loads and Boundary Conditions Subroutines

| | User Required Parameters or Subroutine Model Definition Options | | Purpose |
|--|--|--|--|
| | CREDE | THERMAL LOADS | Definition of state variable including temperature. |
| | CUPLFX | COUPLE DIST FLUXES (flux type 101) | Heat generated due to inelastic behavior in coupled analysis. |
| | DIGEOM | CONTACT (2D) CONTACT (3D) | Definition of rigid surface. |
| | FILM | HEAT or COUPLE FILMS (Model Definition) FILMS (History Definition) | Definition of convective heat transfer coefficient and sink temperature. |
| | FLOW | HEAT CHANNEL | Definition of mass flow rate. |
| | FLUX | HEAT or COUPLE DIST FLUXES (Model Definition) DIST FLUXES (History Definition) | Definition of distributed flux. |
| | FORCDF | FORCDT FIXED DISP or DISP CHANGE | Definition of point load or kinematic boundary condition in a harmonic analysis. |





Example

```
~*******************
     subroutine flux(f,ts,n,time)
     implicit real*8 (a-h,o-z)
c dimensions for FLUX subroutine
     dimension ts(6),n(7),thick(5)
c
c dimension of arrays for passing data to the Electrochemistry Module
     integer ncell, ngridx, ngridy, ngridair, ngridz, ivchange
     integer i, j, k, l, m, ivflag, iu, iv
     integer ngridzi(4), ivflag, nd, nfueltype, nspecies, iflowtype
  initialize parameters
      call ini set(lxcell, lycell, ivflag, lair, lfuel,
     & vtot, itot, ufuel,
     & pin, iflowtype, nfueltype, ngridair,
     6 ncell, ngridx, ngridy, ngridz, nspecies, txyzn, gxyzn, pxyn,
     & up,fuelvxn,tfuelxn,ap,thick,
     & lzcell, airvn, tairn, tairxyn, tfuelxyn)
      nbhb = 999
     FOLLOWING SECTION CALCULATES INTEGRATION POINT TEMPERATURES
     Code is valid for any kind of problem where temperature at
     integration points is available
     Following code is set up for continuum elements (number of layers = 1)
     icode = 9 = temperatures at integration points
           laver = 1
           icode=9
 average temperatures for number of integration points per element
           tt1 = 0.0
c nn= integration point number
           do nn=1, numip
            call elmvar(icode, m, nn, layer, var)
            tmp = var
            ttl = ttl + tmp
           enddo
c run EC code once per increment and update state variables
      call udf qiv(ivflag,itot,vtot,ufuel,nD,ivpower,heattot,
     * iflowtype, nfueltype, nspecies,
    & lxcell, lycell, lzcell,
```





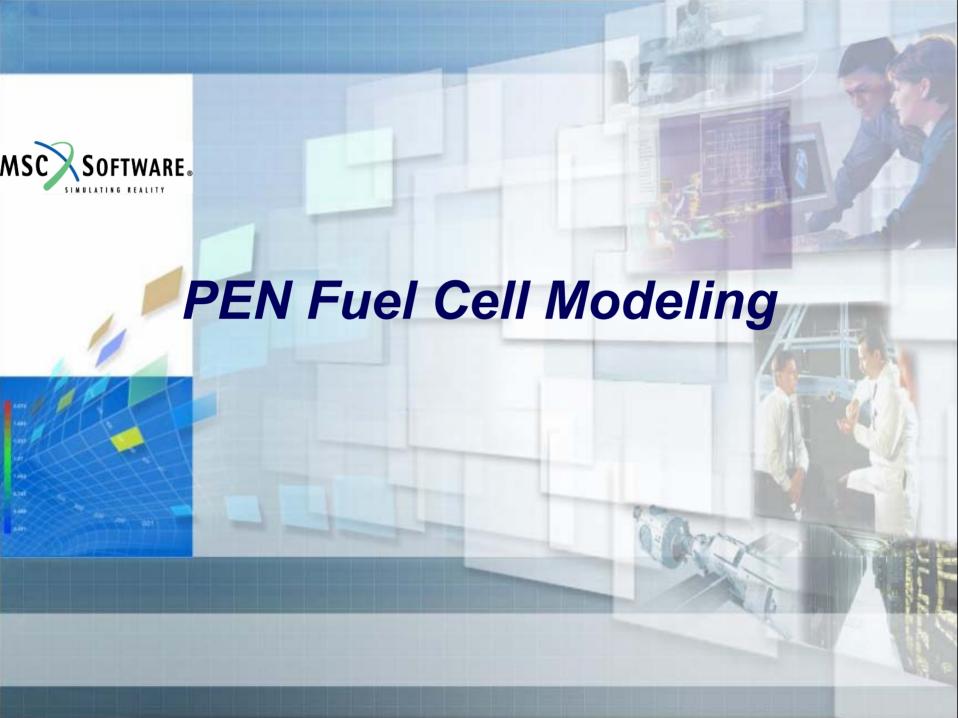


Proposed Development

- To develop a customized graphical user interface for fuel cell stacks.
- To integrate an EC module currently used by PNNL into MSC.Marc to be able to analyze the thermal stresses arising as a consequence of the heat production due to chemical reaction in the fuel cell and heat transfer due to convection effects.
- To conduct a feasibility study of the accuracy and efficiency of MSC.Marc for simulating flow fields in a typical fuel cell stack and to use the results of this feasibility study to determine areas of possible improvement in MSC.Marc's fluid capabilities.







Fuel Cell Layer Configuration

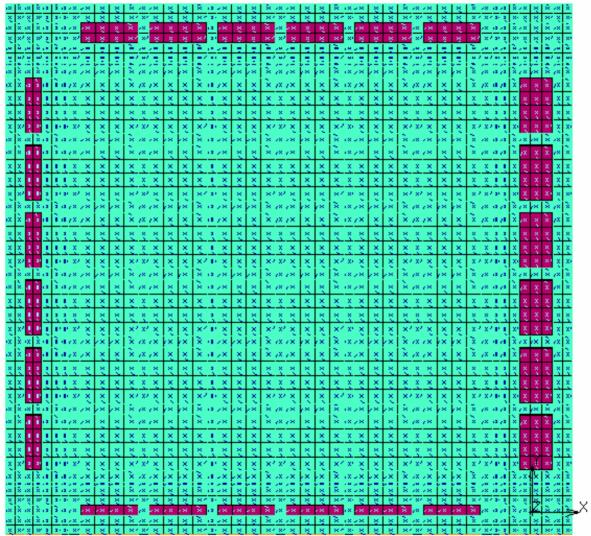
- The following slides show the layers of a typical planar fuel cell design.
- In the example design, the layers stack up to make up air and fuel flow channels on each side of the PEN.
- The PEN has 3 layers, Cathode (air-side), Electrolyte, and Anode (fuel-side). The center area (inside where the PEN seals to the picture-frame) is where the electrochemistry occurs.
- The following slides show "footprints" of the various layers (the selected areas show in the blue-green color).
- The layers are listed in the order from the interconnect (conductive layer between stacked cells), to the cathode-side layers, the PEN, anode-side layers, to the interconnect on the anode-side.
- The last 2 slides show the layers from bottom (cathode) and top (anode) views with the interconnect layers removed.
- The flow/ heat transfer/EC/stress model must have the necessary connectivity through all these layers, including through the flow channels where no mesh is shown in the last 2 slides.





Interconnect



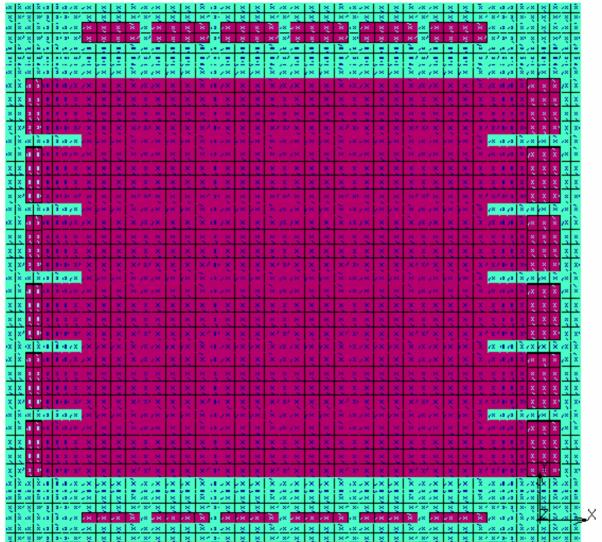






Cathode Spacer



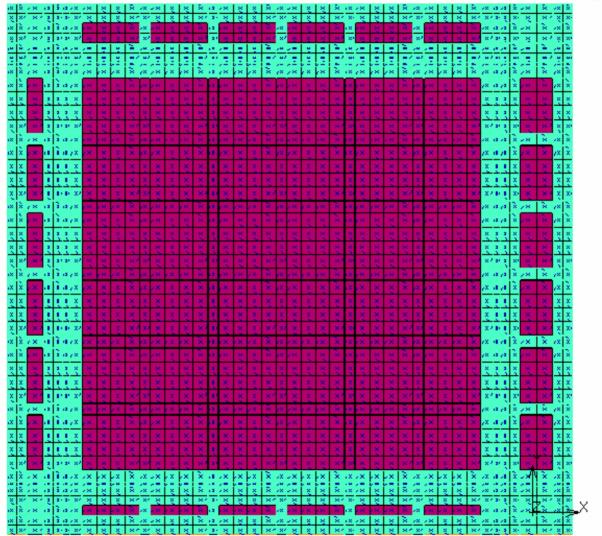






Picture Frame









PEN Inactive Area

MSC

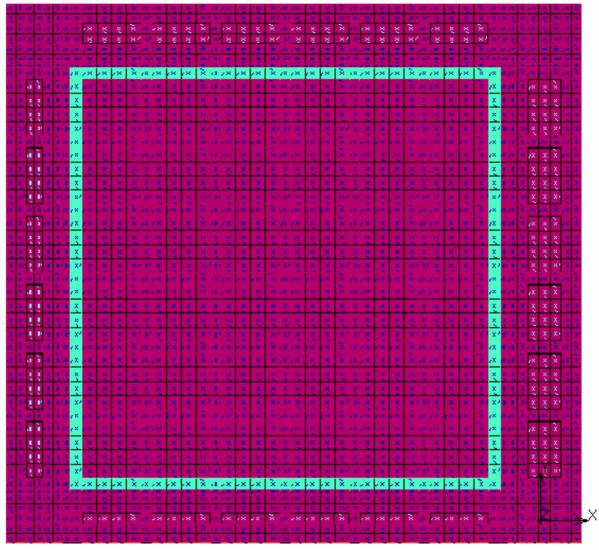
Pen Inactive Area,

Pen-to-Picture frame Seal

(both same Footprint)

Pacific Northwest

National Laboratory





PEN



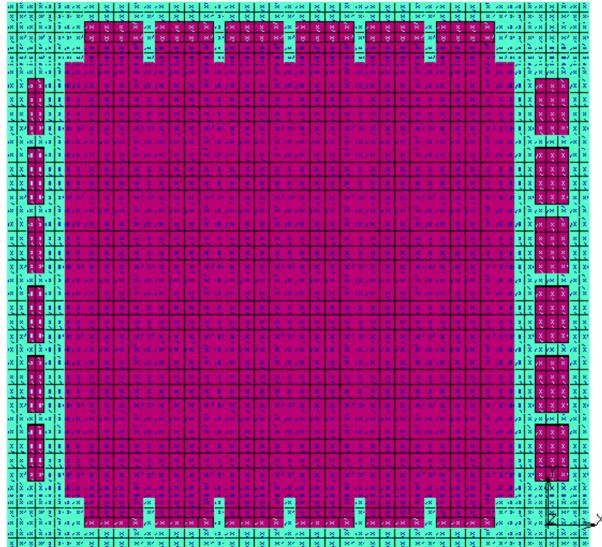
| 2012 2012 3 | i avs vs. | 8 | 8 | S ax ax | 1 18 | 18 | 8 4 | 3 0 M | 8 | × | S . | 8 × 8 | × | × | 8 . | x v× | × | 8 | Se 28 | 200 | × | × | ×. | /X (3 | 23 | žих | 18 | Ř × |
|-------------------|--|-----|------------------|----------|------|-----|-----|---|-------|-----|------|--------|-----|-----|------|---------|-----|-----|--------|------|-----|-----------------------|------|--------|----------------|-------|-------|--------------------------------------|
| 37 S 37 S 3 | 3 18 SE | 8 | × | 80.80 8 | 18 | × | 200 | a+ 8 | 8 | 8 | 207 | ar ag | × | × | 36.5 | 20 E | × | 8 | 20 F 2 | × 8 | × | × | 20.7 | 201.3 | - 3 | 81 X | × | 86.8 |
| X (X X (3) | 1 (1 / 2 / 2 8 | 8 | ×. | 8 20 25 | < ₹ | × | * | 2 ×× | 8 | ×. | × | × 2× | 8 | × | * | × ×× | 8 | 8 | × .* | ××. | ×. | \mathbf{x}^{j} | × | /X (3 | /3 | × ×× | 共 | Si ox |
| 20 8 2 3 | 11 11 11 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 | 8 | ĕ | 84.87.5 | _ | 8 | ** | * * | 8 | ĕ | ** | ×1 8 | 8 | × | ** | x, 8 | 8 | 8 | ×/ × | * | ĕ | × | ×1 | ×2.3 | 3. | 21.3 | × | 27 X |
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| 5.2 5.3 | | Ē. | Ξ | شر سر ج | 1.2 | ĮĒ, | Ξ. | : <u>, </u> | ıΞ | ıΞ | | - , - | Ε. | Ē. | a., | Ξ,Ξ | ıΞ | Ē. | - /- | 12 | Ε. | 3 | | | 21 | 373 | ļΞ | 37.5 |
| X - X - X - X - X | 1 12 /× /× | ٧× | /× | * /× /× | · k× | r×. | * ' | . /× | r×. | ř×. | 187 | × 7× | r×. | /× | 8 1 | × /× | ٧× | × | 3 /× | /× | r×. | × | ×, | | 72 | × /× | /× | × 78 |
| X is a is it | I II /X /X | × | × | × /× /× | ×χ | × | × 1 | L/X | × | × | × / | x yx | × | × | × 1 | x yx | × | × | × /× | /× | × | × | ×, | /X | 78 | X yx | * | × yx |
| 2 2 2 2 1 | X X | × | \mathbf{x}^{j} | × × × | × | × | × | Ų× | × | × | × | ×χ | × | × | × | ž ž | × | × | ×× | × | × | \mathbf{x}^{\prime} | × | X | 1 | X | × | × × |
| 2 2 3 2 3 | • • × × | * | X / | * * * | × | × | × | | * | * | × | * * | * | × 2 | × | | * | * | × × | × | × | x/ | × | × | 3 | × × | 8 | 8 8 |
| X2 2 2 2 1 | 10 X2 X | × | × | ×4 ×4 × | × | × | ×2 | υž | × | × | ×2 | ××. | × | × | ×Z | × × | × | × | ×2.8 | ×ξ | × | \times | X. | X2 | Q. | x × | * | $x\in \chi$ |
| X 100 X 100 3 | 1 12 ×× ×× | × | × | 3 18 18 | : k× | × | × , | 108 | × | × | 3. | * ** | × | × | × , | × > × | × | × | 30.00 | ** | × | × | ×. | ex a | 73 | Řих | jes. | 80 |
| S at it is a | 1 11 /× /× | × | χź | × /× /× | × | × | × · | * /× | × | × | × 7 | ×y× | × | × | × | × /× | × | × | × /× | /× | × | × | × | /× | 72 | ×yx | × | X ex |
| 21111 | X X | × | × | × × × | × | × | × | Ų X | × | × | × | X X | × | × | × | ×× | × | × | ××× | × | × | × | × | X | | X X | × | X X |
| 2111 | X X | × | × | × × × | × | × | × | × | × | × | × | ž ž | × | × | × | X X | × | × | ××× | × | × | \times^{\prime} | × | X | | X X | × | X X |
| 32 t 10 g 3 | ** ×* × | × 2 | X 2 | **** | × | × | × | n 8 | × | × | ** | 87 K | × | × 2 | ×ď | ** * | * | × 2 | ×2.8 | 18 | × | \mathbf{x}^{2} | × 2 | ×2 | e. | xı X | × | X) × |
| \$ 18 \$ 15 B | 1 12 /× /× | × | × | \$ 1× 1× | · k× | × | × , | 1/8 | × | ŀ×. | 87 | × 7× | × | × | × , | × /× | × | × | \$ 7× | 78 | × | × | Э, | /X B | 72 | À /× | post. | 8 7× |
| X 12 2 12 3 | i ii /x /x | × | × | × /× /× | ĺχ | × | ×ο | i yx | × | × | ×, | x /x | × | × | ×, | xyx. | × | × | × /× | /× | × | × | ×, | /X | 71 | x ex | * | × /x |
| 2 2 3 2 3 | 11 8 8 | Z, | χź | * * * | × | × | × | : : | × | × | × | * * | × | × | × | * * | × | X. | × 2 | × | × | χź | × | × | 3 | × × | | 8 8 |
| 2 2 3 2 1 | X X | × | × | × × × | × | × | × | Ų X | × | × | × | ×χ | × | × | × | ž ž | × | × | × × | × | × | × | X | X | Ţ | X X | × . | * * |
| X2 2 2 2 1 | 14 X7 X | ×× | × | ×4.84.8 | × | × | ×Z | υ× | × | × | ×2 | ×× | × | × | хZ | x × | × | ×× | X4.X | Žχ | × | × | X. | X2 | ą. | x: × | , × | 81 X |
| X 18 X 19 3 | 1 12 /× /× | × | × | \$ 7× 7× | · k× | ×. | × . | 178 | × | ŀ×. | i | ×7× | × | × | × , | × /× | × | × | \$ 7× | 78 | × | × | ×, | æ. | 12 | λy× | þ×. | 8 78 |
| Sali o S | i a /× /× | × | × | × /× /× | ١× | × | × , | 1 /× | × | × | × / | × y× | × | × | × | × /× | × | × | 200 | /× | × | × | ×, | es a | e ^a | × ,x | × | X ex |
| 2 () () | X X | × | × | * * * | × | × | × | Ų X | × | × | × | ×χ | × | × | × | X X | × | × | × × | × | × | × | × | X | ţ | X X | × | X X |
| < !!! !! !! | | /X | × | * * * | ŧξ | × | × | ₹ ₹ | × | × | × | * * | × | × | × | * * | × | Z. | * * | × | × | × | × | × | 3 | ŧ X | ×× | X × |
| 22 T P 1 1 | X ² X | × | × | ×2 ×2 × | × | × | ×4 | P × | × | × | × | x² × | × | × | × | x × | × | × | ×2.X | 7 × | × | × | X. | X2 | • | x, X | | $x \in \mathbb{X}$ |
| X ox X or 3 | 1 1 /X /X | ν× | ν× | \$ 1x 1x | · k× | | × . | ιź | ν×. | ĺ×. | 18.7 | × /× | | rΧ | × 4 | x z x | ν× | ν× | \$ 1x | 7× | ٧× | ı× | Σ, | α | ai. | λ'n | įχ | Ř zx |
| ka ka k | i a zx zx | × | × | \$ 1× 1× | ŧİχ | × | ×. | r /× | × | × | ×, | xyx. | × | × | ×. | x /x | × | × | \$ 1x | × | × | × | x, | χ | a | x ex | × | x ex |
| × 1 1 1 1 | | × | × | * * * | į į | × | × | | × | × | × | * * | × | × | ×× | × × | × | × | * * | × | × | × | × | × | 2 | ×× | * | × |
| 2111 | X X | × | × | ××× | × | × | × | į× | × | × | × | χ× | × | × | × | ×χ | × | × | ×× | × | × | × | × | X | į | X X | × | x x |
| X2 1 1 1 1 | X/ X | × | × | X7 X7 X | × | × | ×2 | iοχ | × | × | ×2 | x/ × | × | × | ×2 | x × | × | × | X7.X | × | × | × | X. | X2 | đ | x x | × | X1 X |
| 8 (8 8 (8) | i a /× /× | v×. | ×. | \$ 7× 7× | · k× | ×. | â. | . 🔊 | V×. | Į×. | × , | × 78 | ×. | ×. | â, | × /× | V×. | × | \$ 7× | 18 | ×. | œ. | ×, | e s | 12 | ž ,× | per | $\tilde{\mathbf{x}}_{i,j}\mathbf{x}$ |
| 8 a i a 3 | 1 13 /× /× | × | × | \$ 7×7× | ĺ٠ | × | ×. | | × | × | × / | × 7× | × | × | ×, | × /× | × | × | 20 | /× | × | × | × , | ex s | es. | × ,x | × | x ex |
| x 1 1 1 1 | X X | × | × | × × × | × | × | × | × | × | × | × | ×× | × | × | × | ×× | × | × | ××× | × | × | × | × | × | • | x x | × | x x |
| 2 1 1 2 2 | 2 × × | × | × | * * * | × | × | × | : × | × | × | × | × × | × | × | × | × × | × | × | × × | × | × | × | × | × | 3 | × X | × | X × |
| x2 2 2 3 1 | x' × | × | × | ×1 ×1 × | × | × | × | P X | × | × | × | x' × | × | × | × | x × | × | × | ×2.x | 7 × | × | × | x, | X2 | | x × | į. | ×1 × |
| X .x X . i 3 | i a vx v× | × | ν× | 3 .x .x | i k | × | À. | ωž | ı×. | ŀ×. | À., | x vx | × | × | ×. | xux. | ı×. | × | \$.x | ×x. | × | × | X. | x. | λÌ, | ÌУх | Įλ | $\widehat{X} \cup \widehat{X}$ |
| X X I | 10/20/20 | 8 | × | 5 /2 /2 | 12 | × | ā.; | 152 | - | - E | £ 2 | 8 / B | 8 | × | ā : | 2 J. S. | × | 8 | S /2 | 75 | × | × | Ŧ. | , u | 3 | 575 | 15 | \$ 78 |
| 87 N 2 3 | 3 - 3 - 3 - S | × | × | ×4 ×4 × | 18 | × | 80 | a - 18 | × | × | 26.7 | 84 N | × | × | 24.5 | 80 B | × | × | 84.8 | 1 15 | × | × | 20.7 | 20 ° 3 | Ę | × × | Ė | 84.8 |
| X 18 X 19 3 | 3 0 7 2 2 8 | ××. | e X | 8 10 20 | < ×× | /× | * . | ** | ××. | e X | * , | × 2× | ×× | ×. | 8 | × ×× | /× | × | × /* | ×× | ÞΧ | /× | ¥, | /H (3 | e^{2} | k /× | 1 | 8 /8 |
| X 48 X 43 3 | 100000 | × | × | × 28 28 | 1 18 | × | 8 . | 3 28 | × | × | × / | N AN | × | × | 8 . | N o H | × | × | \$ 2X | 236 | × | 8 | ×. | -X (3 | -3 | ×их | × | 8 28 |





Anode Spacer



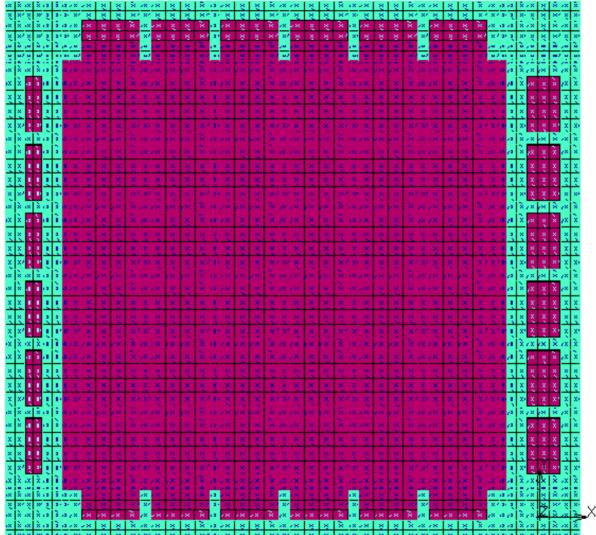






Seal to Interconnect



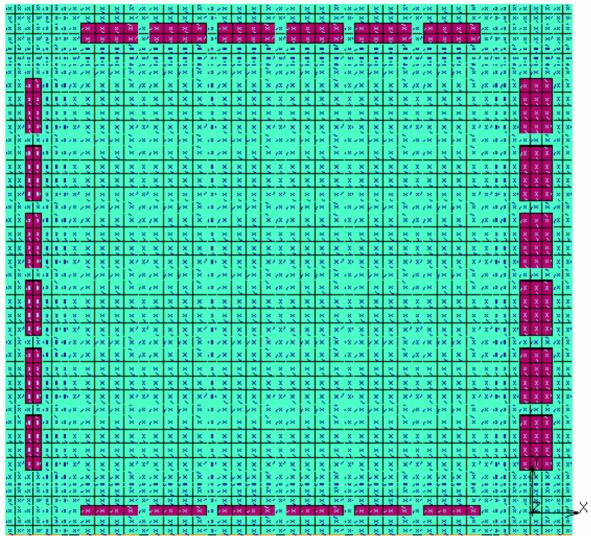






Interconnect

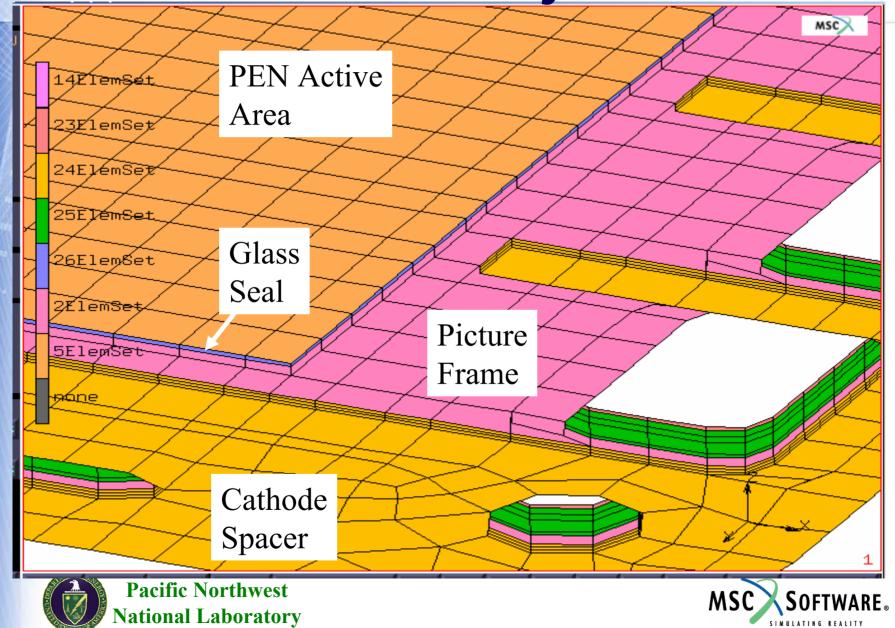




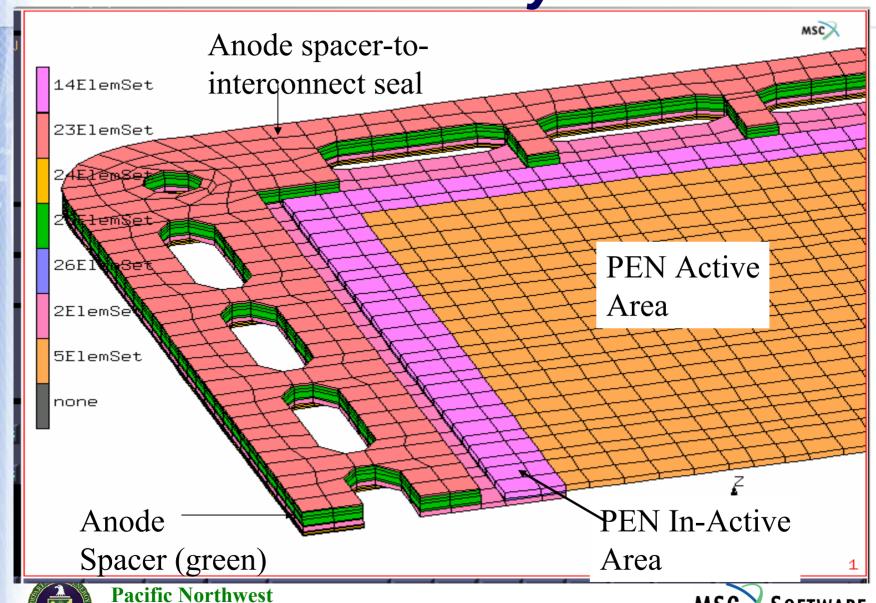




Assembly

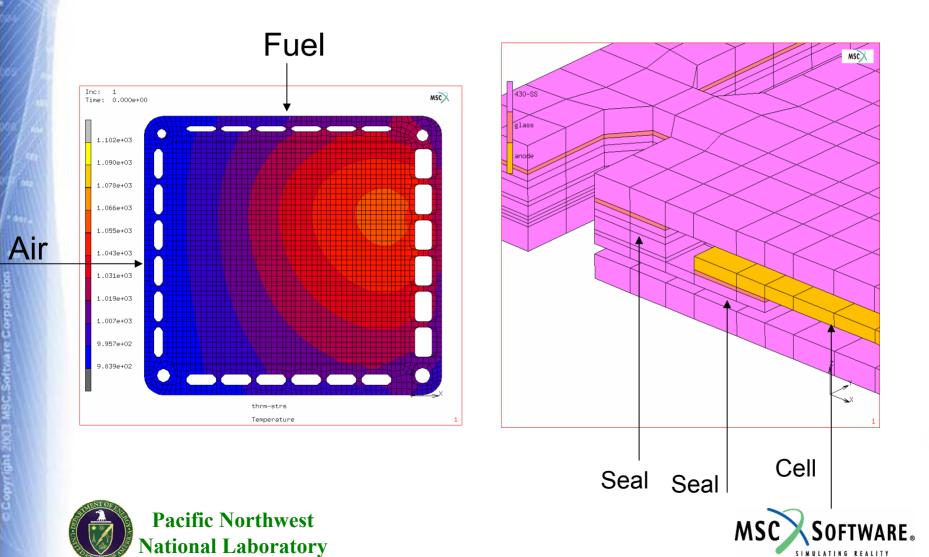


Assembly

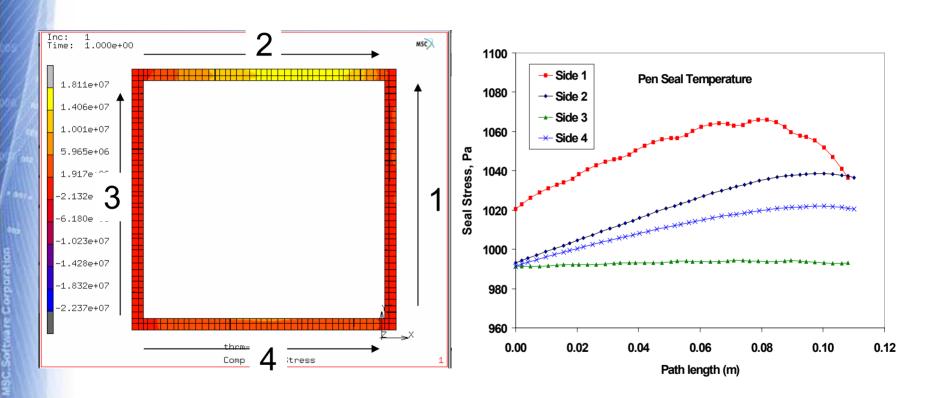


National Laboratory

Fuel Cell Geometry



Seal Temperature Profiles

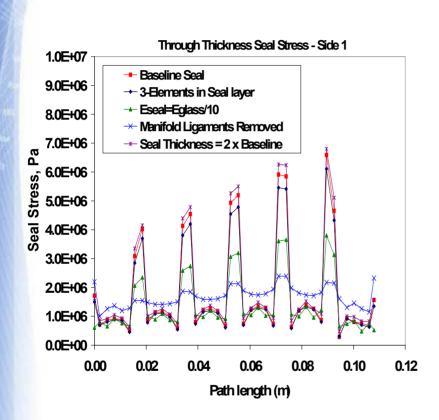


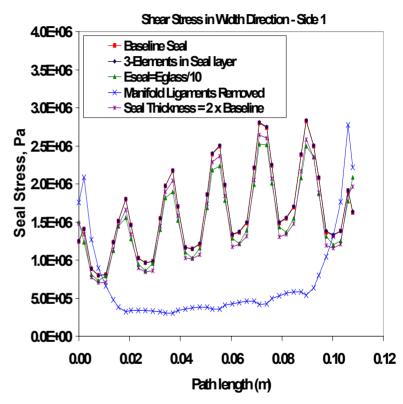
Glass Seal Temperature





Through Thickness Stress



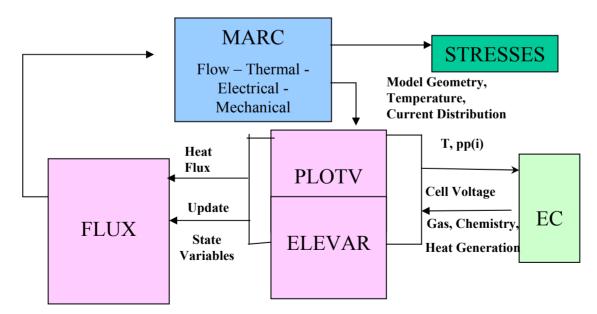






EC in FEA Framework

- Fuel cell operations involve multi-physics processes; the constitutive thermal, chemical, electrochemical and transport processes are strongly coupled => requiring versatile multiphysics tool for realistic description
- Technology development involves design optimization of various geometric, material and operation parameters; the cost of such parametric studies increases exponentially with the number of the working parameters and there are many such parameters involved
- => Computational efficiency is critically important

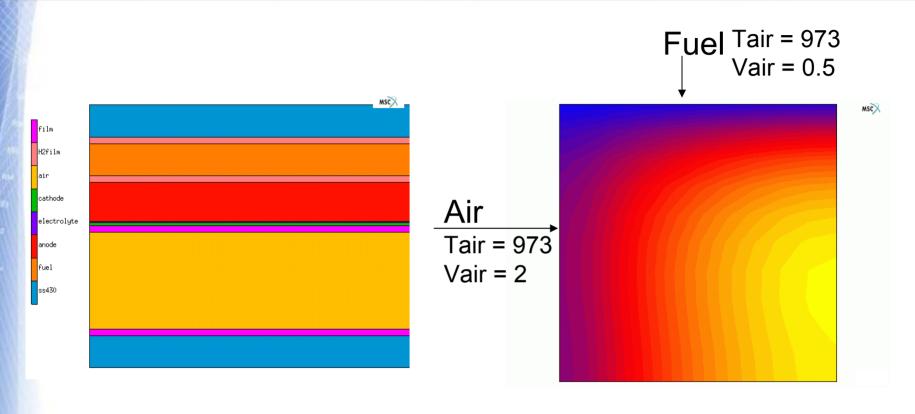






03 MSC.Software Corporation

EC in FEA Framework



Material Layers Thickness 4mm

.11m square



