## Chemical kinetic modeling development and validation experiments for direct fired sCO2 combustor

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Note: This version of the presentation is approved for public release. Please contact <u>subith@ucf.edu</u> for further information.

## Project/Task Summary

Task 1: Project Management

Tasks 2&3: Acquire kinetics and ignition data in highly CO<sub>2</sub> diluted mixtures with shock tube experiments

Task 4: Refine and validate a chemical kinetic mechanism for Supercritical Carbon Dioxide (sCO<sub>2</sub>) Mixtures

Task 5: Develop a CFD Code that utilizes mechanism for sCO<sub>2</sub> combustors

## Motivation

- Current state-of-the-art, such as GRI-3.0 Mechanism, has only been validated for pressures up to 10 atm
- Mechanisms have not been developed for CO<sub>2</sub> diluted mixtures
- Updated mechanism will allow for accurate combustor modeling with multistep combustion using a validated mechanism
- Current CFD combustion models do not consider non-ideal effects



Effects of Increasing Pressure. Equilibrium calculation for  $CH_4/O_2/CO_2$  at  $\phi = 1$ . Figure adapted from Strakey, 2014, sCO2 symposium

## Motivation

Current state-of-the-art differ in their predictions even at atmospheric pressure



- GRI 3.0 is still a widely used mechanism created 15 years ago
- Aramco Mech 1.3 is a recent well-validated mechanism

## Motivation

Effect of CO+OH $\rightarrow$ CO2+H and third-body collision efficiency of CO2 on ignition times



### Existing High-Pressure Methane Ignition Data



 $CH_4/O_2/N_2/Ar$  ignition delay time measurements. The higher pressure data exhibit a significantly weaker variation with temperature (smaller activation energy) than the lower pressure, higher temperature mixtures

JPP, 1999, 15(1), 82-91

## Tasks 2&3: Experimentation

Experiments will be performed in two different shock tube facilities for Methane Oxidation diluted with CO<sub>2</sub>

Experiments will be performed pressures up to 300 bar for temperatures between 800 K and 2000 K and equivalence ratios of 0.7 to 1.2

Ignition delay times and key species time histories will be measured

Experiments will also be performed for selected mixtures of syngas

## Combustion chemistry process



## Shock tube x-t diagram



## Laser absorption spectroscopy



S(T): line strength,  $\Phi(T,P,X)$ : line shape

## Tasks 2 & 3 Sample Results: Shock Tube



Reflected shock wave experiments: sidewall pressure profiles. Reflected shock conditions: 1298K, 61.24atm, 50% CO2/50% argon mixture.

![](_page_11_Picture_2.jpeg)

## Ignition Delay Time Measurements

Ignition delay times measured from the arrival of reflected shockwave to rise of the pressure trace

Arrival of shockwave determined as midpoint of the second pressure rise (rise due to reflected shock)

Rise of OH Emissions measured as the intersection between the baseline and the tangent line drawn from maximum rise of OH

![](_page_12_Figure_4.jpeg)

### Methane Concentration Results

- Comparison of measured and simulated methane concentration for
  - Stoichiometric ignition of 3.5% CH<sub>4</sub> in Argon, 1600K

![](_page_13_Figure_3.jpeg)

## Methane Concentration Results

![](_page_14_Figure_1.jpeg)

- Comparison of measured and simulated methane concentration for
  - Stoichiometric ignition of 3.5% CH<sub>4</sub> in Argon diluted with 30% CO<sub>2</sub>, 1600K

## Methane Ignition and Concentration Results

(Combustion and Flame, Koroglu, Vasu, et al. 2016)

- Comparison of measured and simulated methane concentration for
  - Stoichiometric ignition of 3.5% CH<sub>4</sub> in argon/CO<sub>2</sub>
  - GRI predictions are wrong for both ignition time and concentrations even at low pressures !!

![](_page_15_Figure_5.jpeg)

Methane/O<sub>2</sub>/CO<sub>2</sub> Ignition: High-Speed Imaging for Accurate Ignition Determination

# Methane/O2/CO2 Ignition: High-Speed Imaging Results XCO2=0

![](_page_17_Figure_1.jpeg)

## Methane/O2/CO2 Ignition Imaging Results

![](_page_18_Figure_1.jpeg)

2%CH<sub>4</sub>/2%CO/5%O<sub>2</sub>/85%CO<sub>2</sub> 12 atm

![](_page_19_Figure_1.jpeg)

## Syngas/O2/CO2 Ignition Delay Times

Syngas fuel is a mixture of CO and  $H_2$ 

#### Syngas /O2/CO2 Ignition Delay Time Measurements CO2=80%. Syngas=50% H2+ 50% CO

![](_page_21_Figure_1.jpeg)

#### Syngas /O2/CO2 Ignition Delay Time Measurements CO2=60%. Syngas=80% H2+ 20% CO

![](_page_22_Figure_1.jpeg)

- Combustion kinetics model refinement/development
- Existing kinetic models are only valid at low pressures < 50 atm</li>
- We will use multiscale simulations to extend their validity to above 300 bar by:
  - **1.** Quantum Mechanic simulations of the activation enthalpies in gas vs. CO<sub>2</sub> environment
  - 2. Molecular Dynamic simulations of reaction processes

## Combustion chemistry/kinetics

![](_page_24_Figure_1.jpeg)

- A rate coefficient must be prescribed for each chemical reaction
- k (T) = A x exp(- $E_a/RT$ ), T: temperature,  $E_a$ : activation energy, R: gas constant
- Rate coefficients can be measured using advanced laser diagnostics
- Models need to be reduced based on experience and intuition so they can be used with CFD

## The important elementary steps in RD2010 mechanism of C0-C4 fuel combustion

Table 4 Important reactions and the source of their rate constants used in the mechanism. Rate coefficients are in units of cal, mol, cm<sup>3</sup>, K. Rate constants are calculated as  $k = A^*T^n * Exp(-Ea/RT)$  where T represents temperature and R represents the gas constant.

No.	Reaction	А	n	Ea	Ref.
1	$H + O_2 = O + OH$	3.55E+15	-0.406	1.66E+04	[7]
2°	$CO + OH = CO_2 + H$	2.20E + 05	1.89	-1.16E+03	[7], A*1.24
3	HCO+M = H+CO+M	4.75E+11	0.7	1.49E + 04	[6] <sup>a</sup>
4	$H+OH+M=H_2O+M$	4.50E + 22	-2	0.00E + 00	$[19]^{a}$
5	$C_{3}H_{5}-a+H(+M) = C_{3}H_{6}(+M)$	2.00E + 14	0	0.00E + 00	[8]
	Low pressure limit:	1.33E + 60	-12	5.97E+03	
	Troe parame	eters: 0.02, 1.10E+03, 1.10	)E+03, 6.86E+03		
6	$CH_3 + CH_3(+M) = C_2H_6(+M)$	9.21E+16	-1.17	6.36E+02	$[7]^{a}$
	Low pressure limit:	1.14E + 36	-5.246	1.71E + 03	
	Troe para	meters: 0.405, 1.12E+03,	69.6, 1.00E+10		
7	$CH_3 + HO_2 = CH_3O + OH$	1.00E + 12	0.269	-6.88E+02	[7]
8	$CH_4+H=CH_3+H_2$	6.14E + 05	2.5	9.59E+03	[7]
9	$HO_2 + HO_2 = H_2O_2 + O_2$	4.20E + 14	0	1.20E + 04	[18] <sup>b</sup>
		1.30E+11	0	-1.63E+03	
10	$CH_4 + HO_2 = CH_3 + H_2O_2$	1.13E+01	3.74	2.10E + 04	[7]
11	$CH_3O_2+CH_3=CH_3O+CH_3O$	5.08E+12	0	-1.41E+03	[7]
12	$CH_3 + OH = CH_2(S) + H_2O$	4.51E+17	-1.34	1.42E + 03	[7]
13	$CH_3+O_2=CH_2O+OH$	2.64E + 00	3.283	8.11E+03	[7]
14	$CH_3 + H(+M) = CH_4(+M)$	2.14E+15	-0.4	0.00E + 00	$[1]^{a}$
	Low pressure limit:	3.31E+30	-4	2.11E+03	
	Troe pa	rameters: 0.0, 1.00E-15, 1	.00E-15, 40.0		
15 <sup>°</sup>	$C_2H_4 + H(+M) = C_2H_5(+M)$	1.95E + 12	0.454	1.82E + 03	[ <b>7</b> ] <sup>a</sup> , A*1.8
	Low pressure limit:	2.16E + 42	-7.62	6.97E+03	
	Troe p	arameters: 0.975, 210, 984	4, 4.37E+03		

<sup>a</sup>Collision efficiencies: CH<sub>4</sub> 2.0, CO 1.9, CO<sub>2</sub> 3.8, C<sub>2</sub>H<sub>6</sub> 3.0, H<sub>2</sub>O 6.0, H<sub>2</sub> 2.0, Ar 0.7.

<sup>b</sup>Rate constant is the sum of two expressions.

Naik, C. V.; Puduppakkam, K. V.; Meeks, E. *J Eng Gas Turbines Power* **2012**, 134, 021504.

#### Gas-Phase Combustion Reaction $H+O_2 \rightarrow O+OH$

- Five distinct steps along Reaction (
  - 1. Reactants (R1,R2)
  - 2. Reactive complex (RC)
  - 3. Transition state (TS)
  - 4. Product complex (PC)
  - 5. Products (P1,P2)
- Two radicals may couple high spin corresponding to two multiplicities
- QST3 method will be used to locat
- IRC method will be used to connec

![](_page_26_Figure_10.jpeg)

#### Gas-Phase Heat release Reaction $CO+OH \rightarrow CO_2+H$

![](_page_27_Figure_1.jpeg)

Pathways shown for gas phase

The supercritical solvent can modify predictions of this model in three ways:

- changing the ability to reach the equilibrium by the reactants and/or TS
- shifting this equilibrium, and
- changing probability of TS to convert to the products

Both factors will be studied using Quantum Chemistry methods:

- Changing reaction mechanism (CO<sub>2</sub> may be involved in TS structure)
- Products and TS may be stabilized by CO<sub>2</sub> to a different degree (thus, changing ΔH<sup>≠</sup>)

#### Task 4 Sample Results: Quantum Mechanical Calculations

Elementary Reaction CO+OH $\rightarrow$ CO<sub>2</sub>+H (results with covalent CO<sub>2</sub> addition: new mechanism discovered !!)

![](_page_31_Figure_1.jpeg)

Figure 2. Relative energies (kcal/mol) of the reaction pathway shown on Scheme 1, with one covalently bound  $CO_2$  molecule (*trans*-HOCO +  $CO_2$  system is chosen as the reference point).

Elementary Reaction  $CO+OH\rightarrow CO_2+H$ (results with covalent  $CO_2$  addition)

![](_page_32_Figure_1.jpeg)

Elementary Reaction CO+OH $\rightarrow$ CO<sub>2</sub>+H (results with spectator CO<sub>2</sub> molecule)

![](_page_33_Figure_1.jpeg)

Figure 3. Relative energies (kcal/mol) of two of the reaction pathways shown shown on Scheme 2 (in black) and Scheme 3 (in red) with spectator  $CO_2$  molecule (the *trans*-HOCO +  $CO_2$  system is chosen as the reference point).

### Elementary Reaction CO+OH $\rightarrow$ CO<sub>2</sub>+H (results)

![](_page_34_Figure_1.jpeg)

- Pathways shown for CO2 autocatalytic effect published in J. Phys Chem A- Masunov & Vasu (2016)- above work
- CO2 opens up new pathways and accelerates heat release
- Similar catalytic effects by CO2 seen in other reactions  $(HO_2+HO_2 \rightarrow H_2O_2 + O_2)$  but not in  $H_2CO_2 + HO_2 \rightarrow HCO + H_2O_2$

# Task 5: CFD development and implementation in OpenFOAM

- Real Gas Equations of State for sCO2
- Thermal properties for sCO<sub>2</sub> combustor
- CFD simulation status
- Next steps

Real Gas Equations of State for sCO2 For density and reaction rates

- EOS is a relation between Temperature, Pressure and Density of a system.
- Estimated operating conditions of sCO2 combustor:

![](_page_37_Figure_3.jpeg)

![](_page_38_Figure_0.jpeg)

Figure 1. Classification of various type of equations of state, with a selection of equations for each group. In this classification, van der Waals EoS are those cubic and non-ubic equations that consider the compressibility factor as  $Z = Z^{rep} + Z^{att}$ .

Simple algebraic equations; not as accurate, but popular because they are quick to solve.

EoS's are for single species, need relationships for mixture properties

#### REFPROP

REFPROP is a program that uses equations for the thermodynamic and transport properties to calculate the state points of the fluid or mixture. These equations are the most accurate equations available world wide.

→ Most accurate EOS equations available, but limited pressure and temperature ranges.

→ No data is available for many combustion species.
 Example:
 Species that are common between REFPORP and GRI30 are only:
 H2O; H2; O2; CO; CO2; CH3OH; C3H4; CH4; C2H6; C3H8; C3H6

 $\rightarrow$  REFPROP is reported to be very time expensive for sCO2 simulations. (Mark Anderson et. al., 2016)

• van der Waals Equation of state:

$$p = \frac{RT}{V_m - b} - \frac{a}{V_m^2}$$

- b corrects for the volume available for random molecule movement
- A reduces kinetic energy due to intermolecular forces
- If first term dominates, Z>1 and rho less than ideal gas rho.
- If second term dominates, Z<1 and rho greater than ideal gas rho.
- van der Waals has poor accuracy, but is instructive.

• Peng-Robinson Equation of state:

$$p = \frac{RT}{V_m - b} - \frac{a\alpha}{[Vm(Vm + b) + b(Vm - b)]}$$
Where,  

$$a = \frac{0.45724 R^2 T^2_c}{P_c}$$

$$b = \frac{0.07780 RTc}{P_c}$$

$$k = \frac{0.07780 RTc}{P_c}$$

$$k = \frac{0.07780 Pr}{T_c}$$

$$k = \frac{0.07780 Pr}{Tr}$$

$$k = \frac{0.07780 Pr}{Tr}$$

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$$k = \frac{0.07780 Pr}{Tr}$$

$$\omega - Ancentric factor$$

 $\boldsymbol{\alpha} = [1 + (0.37464 + 1.54226\boldsymbol{\omega} - 0.2699 \boldsymbol{\omega}^2)(1 - \text{Tr}^{0.5})]^2$ 

• RK (Redlick-Kwong), SRK (Soave-Redlick-Kwong) and PR (Peng-Robinson) models are compared against the REFPORP for sCO2

![](_page_42_Figure_2.jpeg)

Note: REFPROP data for CO2 is available 1100 K only.

• RK, SRK and PR models are compared against the REFPORP for sCH4

![](_page_43_Figure_2.jpeg)

Note: REFPROP data for CH4 is available 625 K only.

 RK, SRK and PR models are compared against the REFPORP for sO2

![](_page_44_Figure_2.jpeg)

 $\rightarrow$ None of the models are predicting the sO2 behavior. Possible reasons for this deviation is under investigation.

 $\rightarrow$  'O2' is second largest concentration in the sCO2 combustor.

• PR models are compared against the REFPORP <u>mixtures</u> properties at the <u>Inlet</u> of sCO2 Combustor

![](_page_45_Figure_2.jpeg)

Inlet cond		
Ratio of	Inlet	Exit
CO2/(CH4+O2)	Temperature	Temperature
6.50	500 K	1200 K
7.42	600 K	1200 K
9.15	700 K	1200 K
11.48	800 K	1200 K
15.02	900 K	1200 K

![](_page_45_Figure_4.jpeg)

Note: The percentage differences are averaged over pressures between 30-480 bar at every temperature.

• PR models are compared against the REFPORP <u>mixtures</u> properties at the <u>Outlet</u> of sCO2 Combustor

![](_page_46_Figure_2.jpeg)

Thermal properties for sCO<sub>2</sub> combustor

#### • Compressibility factor (Z) in Real gases

![](_page_48_Figure_2.jpeg)

• 'Z' is related to many thermodynamic relations of Real gases.

Example:

Property	Definition	Ideal Gas Case
Compressibility factor	Z=f(T,p)	Z = 1
Isothermal compressibility	$eta_T = -rac{1}{v} \left( rac{\partial v}{\partial p}  ight)_T = rac{1}{p} - rac{1}{Z} \left( rac{\partial Z}{\partial p}  ight)_T$	$\beta_T = \frac{1}{p}$
Isobaric compressibility	$eta_p = rac{1}{v} \left( rac{\partial v}{\partial T}  ight)_p = rac{1}{T} + rac{1}{Z} \left( rac{\partial Z}{\partial T}  ight)_p$	$eta_p = rac{1}{T}$

Ref: Nikola D. Baltadjiev (2012, MIT Thesis)

#### More thermal properties for sCO<sub>2</sub> combustor Compressibility factor 'Z' under various operating conditions:

Operating Condition	What it explains?	Initial molar mixture and Temperature (K) (CH4/O2/CO2/T)
OP1	Reference Mixture	1/2/24/1000
OP2	Inlet [CO <sub>2</sub> ] increases	1/2/40/1000
OP3	Inlet [CH <sub>4</sub> +O <sub>2</sub> ] increases	2/4/24/1000
OP4	Inlet temperature decreases	1/2/24/800

![](_page_49_Figure_2.jpeg)

#### More thermal properties for sCO<sub>2</sub> combustor Specific heats and ratio of specific heats:

![](_page_50_Figure_1.jpeg)

• The  $c_p$  ranges from 1.331-1.381,  $c_v$  ranges from 1.111-1.187 and  $\gamma$  ranges from 1.2-1.164 across the sCO<sub>2</sub> combustor.

## More thermal properties for sCO<sub>2</sub> combustor Speed of Sound:

![](_page_51_Figure_1.jpeg)

Repulsive forces also results in increasing speed of sound by a factor

$$\left(\frac{\gamma}{n_s Z}\right)^{1/2}$$

#### **CFD** simulation status

![](_page_53_Figure_0.jpeg)

• Further validations with RANS and LES simulations are under progress.

# 4 journal papers and several conference presentations in 2016

- Acknowledgement: DE-FE0025260 (Dr. Seth Lawson as program manager)
- Thank you, Questions?