

# Granular Flow in a Rough Annular Shear

## Validating DEM Simulations with Experiments

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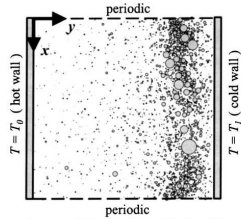


- 1 Background
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  - Discrete Element Method
- 2 Model System
  - Geometry and Materials
  - Geometry and Models
- 3 Results
- 4 Outlook

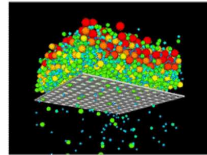


# DEM: the Gold Standard

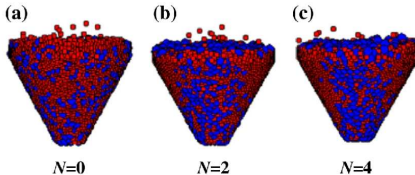
- Model diverse particles and properties
- Measure relevant quantities
- Control material properties
- “Combinatorial” experiments



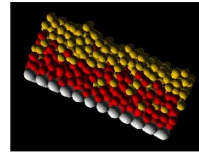
Dahl and Hrenya, Phys. Fluids, 2004.



Clear and Sawley, Appl. Math. Mod., 2002.



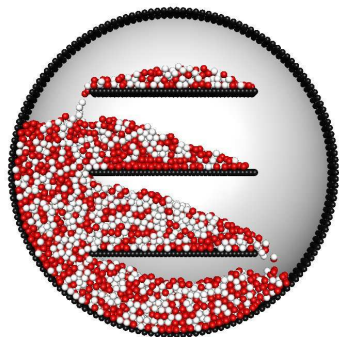
Arratia et al., Pow. technol., 2006.



Khakhar et al., Phys. Fluids, 1997.



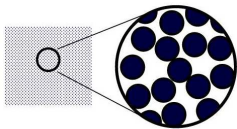
# DEM: the Gold Standard (cont.)



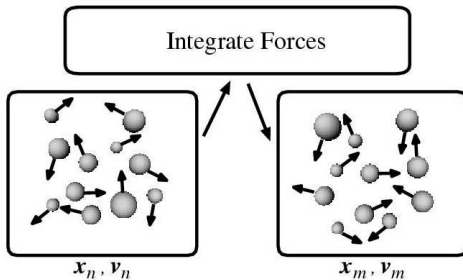
- Remarkable qualitative agreement
- Good macroscopic quantitative agreement (IS, etc.)



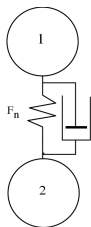
# Discrete Element Method



- Goal: gain *macroscopic* insight from *microscopic* considerations
- Method: Model interaction forces
- Specifics: Newton's Law ( $\mathbf{F} = m\mathbf{a}$ )



# Contact Mechanics – Normal Force Models

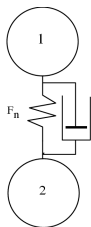


- Simple spring-dashpot model schematic (shown)
- Force models vary in both accuracy and computational difficulty.

Model	Restitution Coefficient (RC)	Mathematical Form	Comments
Purely Viscous (PV : Lee and Herrmann 1993)	Increases with velocity	$k_n \alpha^{3/2} - k_d v_n$	Computationally simple, yet poor RC, discontinuous force vs. approach
Oden-Martins (OM : 1984)	agrees w/experiment	$k_n \alpha^{3/2} - k_d v_n \alpha$	More computationally complex, realistic RC and force vs. approach
Tsuji (T : 1993)	Constant	$k_n \alpha^{3/2} - \bar{k}_d (\sqrt{mk_n}) v_n \sqrt{\alpha}$	More computationally complex, yet yields constant RC and unrealistic force at small unloading
Walton-Braun dependent (WB-d : 1986)	agrees w/experiment	$k_1 \alpha$ $k_2 (\alpha - \alpha_o)$ $k_u = \mathcal{F}(f_n)$	Computationally simple, realistic RC and force vs. approach
Walton-Braun independent (WB-i : 1986)	constant	$k_1 \alpha$ $k_2 (\alpha - \alpha_o)$ $k_2 = \beta k_1$	Computationally simple, constant RC and realistic force vs. approach



# Contact Mechanics – Normal Force Models

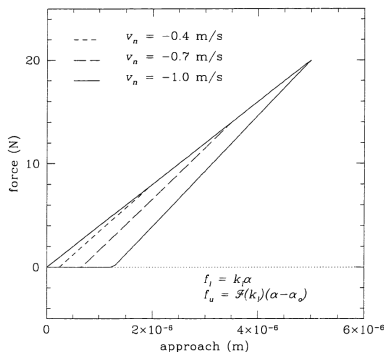
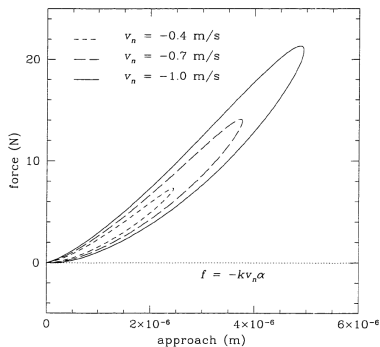


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# Force versus Approach

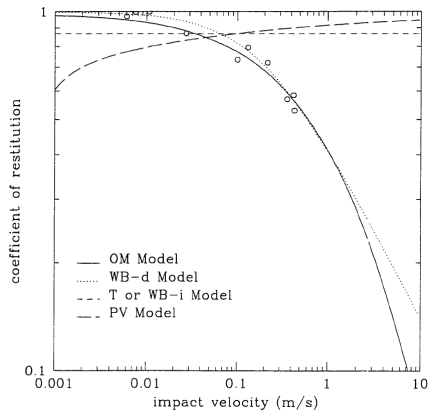


- A simple test of a model's accuracy
- Area “under” the curve represents energy dissipation.





# Coefficient of Restitution



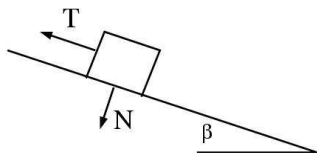
- A useful test of model's dynamic response (typically **only** test)
- Note that CR accuracy is not necessary for obtaining correct kinematics!



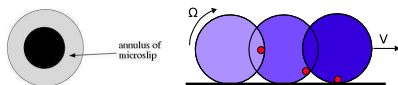
# Contact Mechanics – Friction Forces

Coulomb limit applies after (macro-)sliding occurs:

$$T = \mu N$$



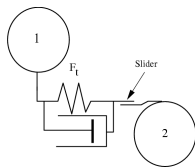
Sliding onset is more complicated (Mindlin, 1949):



- Friction has a “memory”:  $T = T_{old} + k_T s$
- Watch out for rolling on perfectly smooth surface! (rolling friction?).



# Friction Force Models



- Key issue is incremental friction (proportional to displacement, not velocity → creep!)
- Capturing microslip not generally considered critical.

Model	Form	Displacement	$k_t$	Comments
Zero Model (Tsuji 1993)	$-k_t s$	$s = v_t \Delta t$	constant	Computationally simple, yet allows particle creep
One Model (Cundall and Strack 1979)	$-k_t s$	$s = \int_0^t v_t(\xi) d\xi$	constant	More computationally complex, realistic collisions, no particle creep (save $s_n$ )
Two Model (Walton and Braun 1986)	$-k_t s$	$s = \int_0^t v_t(\xi) d\xi$	$k_t = \mathcal{F}(f_t)$	More computationally complex, realistic collisions, no particle creep, dissipates energy through microslip (save $s_n, f_n$ )

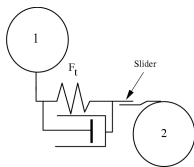
Walton-Braun  
(Two Model):

$$k_t = k_{to} \left( 1 - \frac{f_t' - f_t^*}{\mu f_n - f_t^*} \right)^n \text{ for loading}$$

$$k_t = k_{to} \left( 1 - \frac{f_t^* - f_t'}{\mu f_n + f_t^*} \right)^n \text{ for unloading}$$



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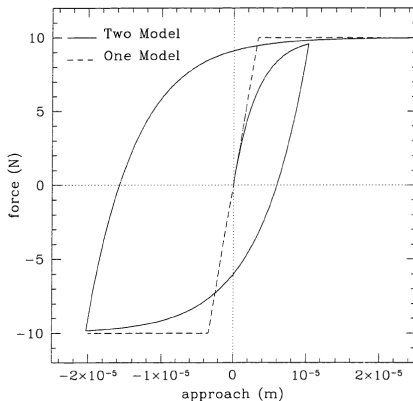
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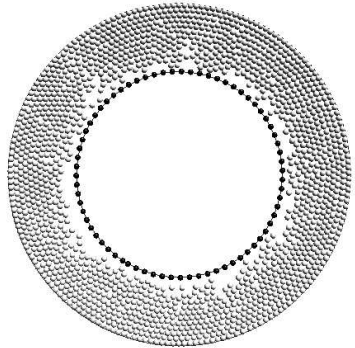
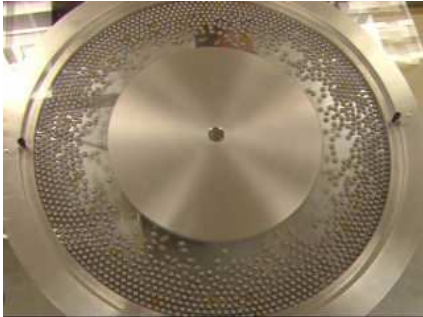
# Force versus Approach



- Note the asymptote to Coulomb sliding
- Zero model not shown since force is not a function of displacement



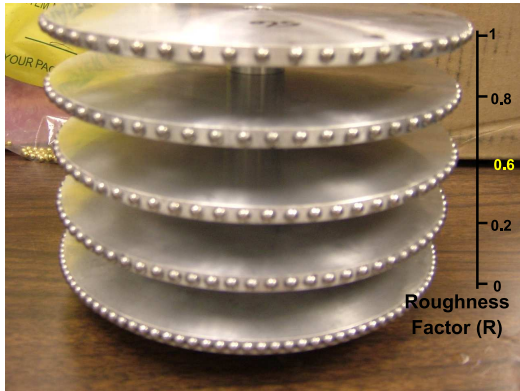
# Model System



- Roughened inner cylinder rotates
- Experimentally extract  $f$ ,  $v$ ,  $T$  profiles



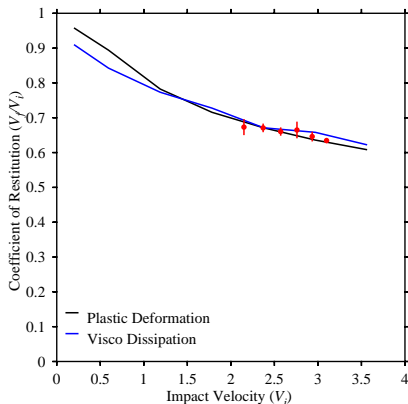
# Model System (cont.)



- Roughness varies from 0→1
- $\Omega$  varies from 220RPM→270RPM
- “Base case”:  $\Omega = 240\text{RPM}$ ,  $R = 0.6$



# Match Properties



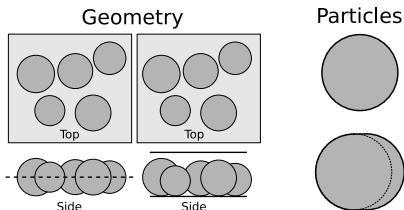
- Match dissipation for both plastic and visco
- Some simulations in 2d, others with varying gaps



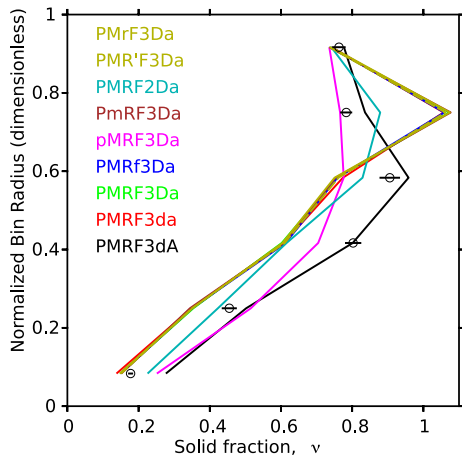


# Geometry and Models

Model variation	Version 1	Version 2+
Normal force model	Plastic [P]	Spring-Dashpot [p]
Friction force model	Mindlin [M]	Cundall [m]
Rolling friction	Large [R']	Present [R] Absent [r]
Dissipation	Fit to experiment [F]	Larger than physical [f]
Geometry	Fit to experiment [3d]	Larger head space [3D] Ideal two dimensional [2D]
Particle Geometry	Aspherical [A]	Perfect spheres [a]



# Solid Fraction by Model

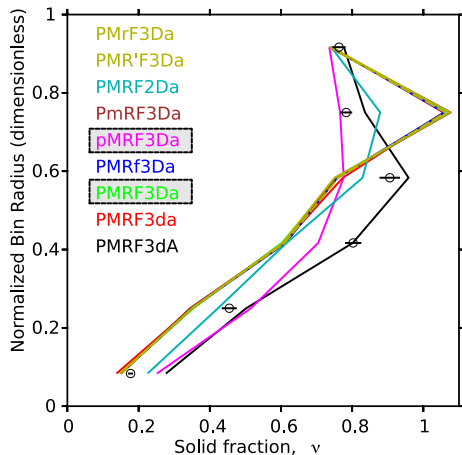


Plastic Dissipation [P]	Spring-Dashpot [p]
Two Model Friction [M]	One Model [m]
Rolling friction [R'/R]	Absent [r]
Dissipation Fit [F]	Larger [f]
Geometry Fit [3d/3D]	Planar [2D]
Particles Aspherical [A]	Spheres [a]

- In 2d, max packing 0.91; 3d systems overlap so  $f$  above 1
- Particle geometry is important; little else matters



# Solid Fraction by Model



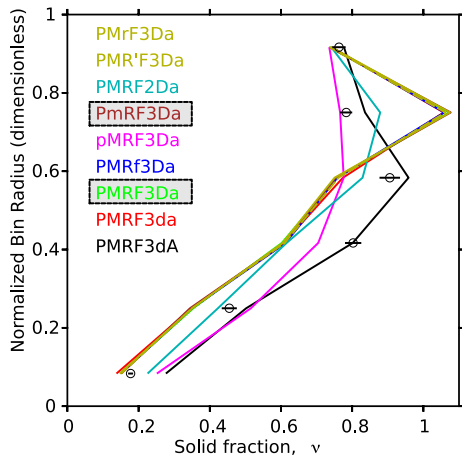
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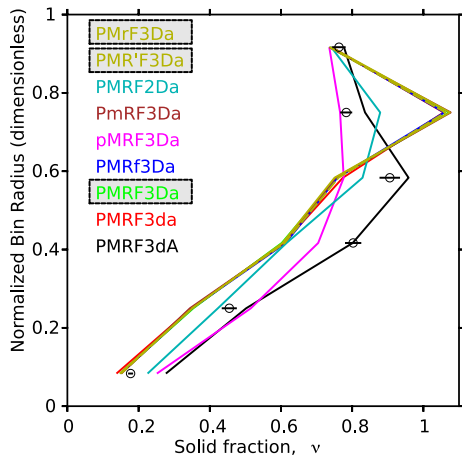


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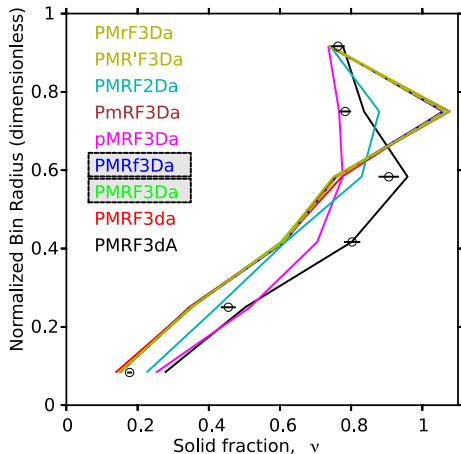


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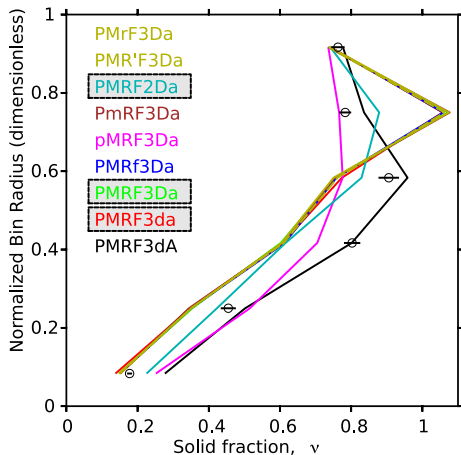


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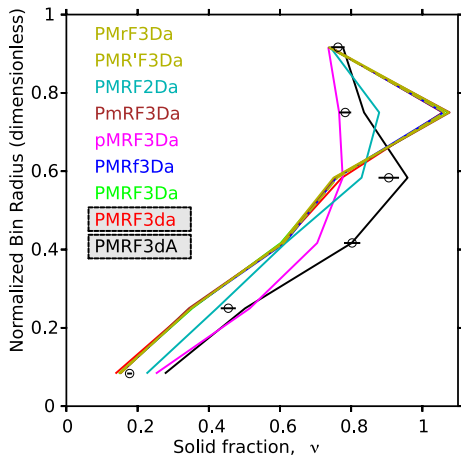


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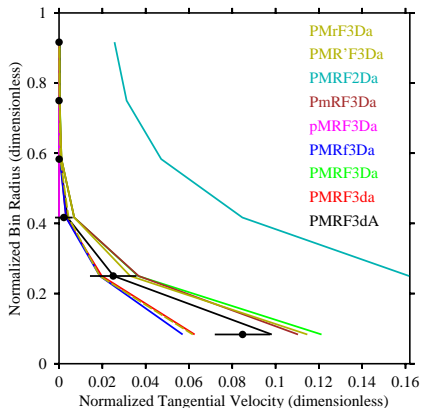
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# Velocity Profile by Model



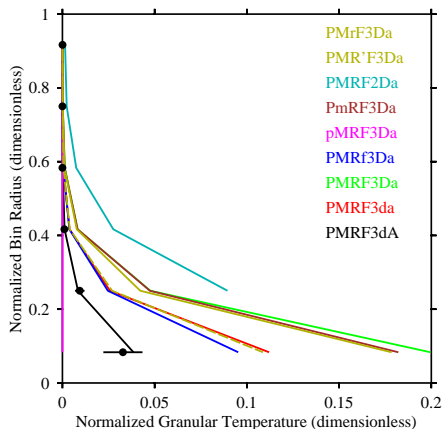
Plastic Dissipation [P]  
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Spring-Dashpot [p]  
One Model [m]  
Absent [r]  
Larger [f]  
Planar [2D]  
Spheres [a]

- System geometry match is *critical* (2D qualitatively wrong)!
- Rolling friction and/or dissipation may be tuned (to mimic asphericity?)
- Visco is way off



# Granular Temperature by Model



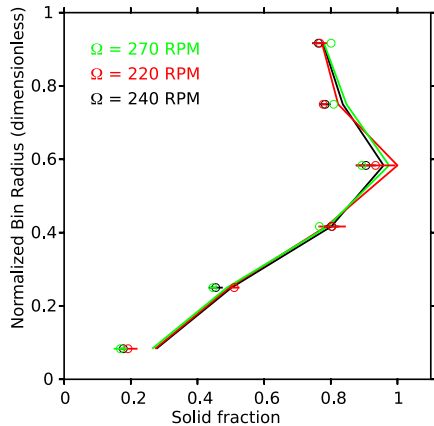
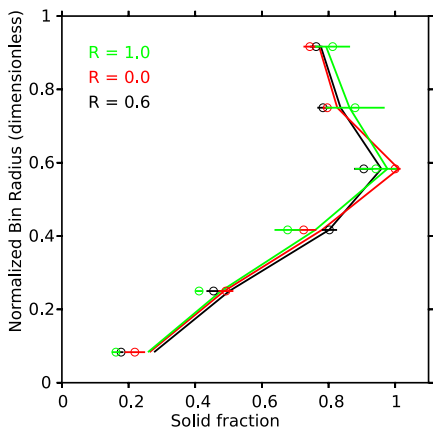
Plastic Dissipation [P]  
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One Model [m]  
Absent [r]  
Larger [f]  
Planar [2D]  
Spheres [a]

- “Extra” dissipation may work (but may create more errors)
- Rolling friction *cannot be tuned* properly
- Visco is way off



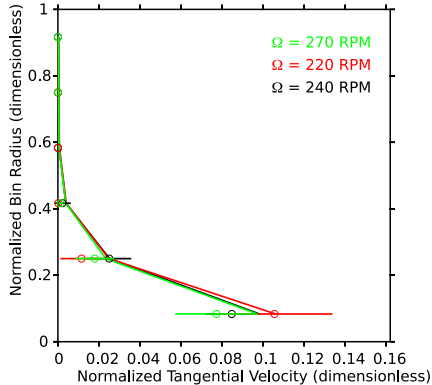
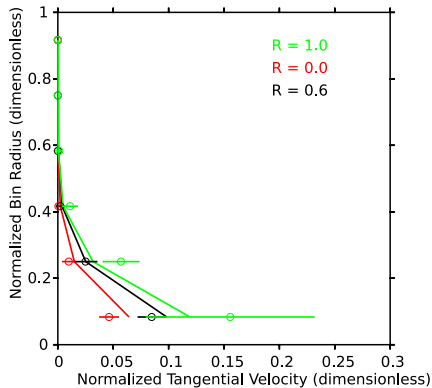
# Varying Roughness/Rotation Rate ( $f$ Profile)



- Max location is very robust
- Roughness simulations captures trends properly (even cross-over)
- Rotation rate is very slightly off



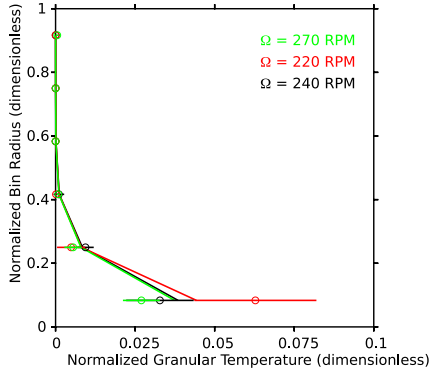
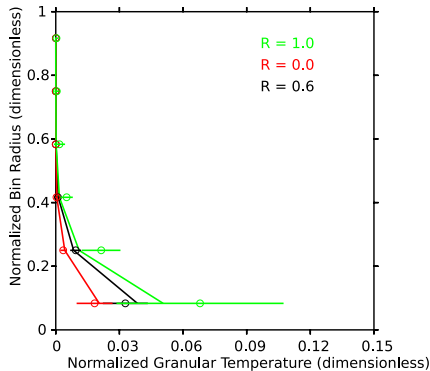
# Varying Roughness/Rotation Rate (Velocity)



- Roughness trend is captured
- Rotation trend is captured



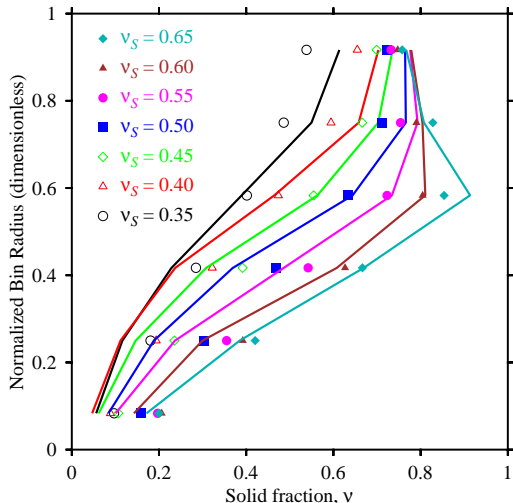
# Varying Roughness/Rotation Rate (Granular Temp)



- Roughness trend is captured
- Rotation rate trend is captured



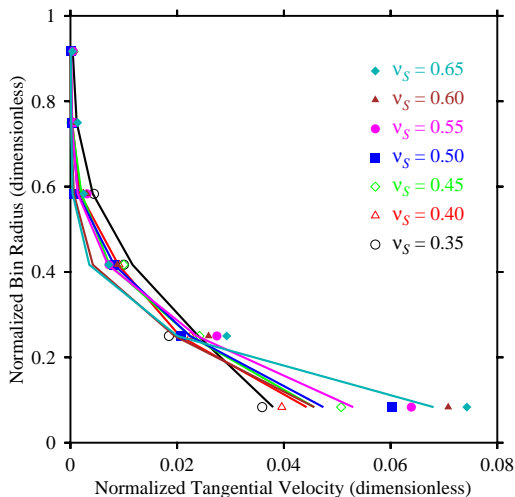
# Parametric Study, $f_{tot}$ (Solid Fraction Profile)



- Surprising agreement both qualitative and quantitative



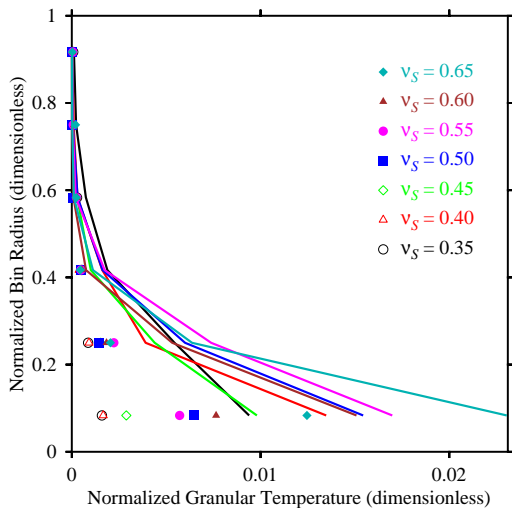
# Parametric Study, $f_{tot}$ (Velocity Profile)



- Qualitative trends are captured
- Slightly off quantitatively (perhaps)



# Parametric Study, $f_{tot}$ (Granular Temp Profile)



- Qualitative trend is decent
- Consistently overpredict T





- DEM “gold standard” – good quantitative
- Modeling **exact** physical geometry is critical
- Modeling of normal force/dissipation is important for  $v$  profile
- Modeling friction is more flexible (likely **not viscous**)
- Rolling friction can compensate for shape for  $v$  or  $T$ , not both
- Particle shape itself needed to capture both  $v$  and  $T$
- Looking at  $f$ ,  $v$ , and  $T$  is surprisingly discriminatory
- Single-particle tests may not tell whole story ...
- Acknowledgment: National Energy Technology Lab, Department of Energy

